

# Biosystems Engineering

## Estimation models for the aerodynamic characterisation of insect-proof screens from their geometric parameters --Manuscript Draft--

<b>Manuscript Number:</b>	YBENG-D-19-00039R2
<b>Article Type:</b>	Research Paper
<b>Keywords:</b>	insect-proof screen; aerodynamic characterisation; aerodynamic model.
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<b>Manuscript Region of Origin:</b>	Europe
<b>Abstract:</b>	<p>This study characterised the geometric and aerodynamic parameters of 35 insect-proof screens with different weft and warp threads, with porosities ranging from 0.237 to 0.556 m<sup>2</sup> m<sup>-2</sup>. The geometric parameters were assessed by analysing digital images, and the aerodynamic parameters were determined using tests in a low-speed wind tunnel. Using the experimental measurements, four different models were developed and validated to estimate the aerodynamic parameters of an insect-proof screen from two or more of their geometric parameters: (i) to estimate the pressure drop coefficient <math>F\phi</math> based on the thread diameter Reynolds number (<math>Re_d</math>) and screen porosity <math>\phi</math> [m<sup>2</sup> m<sup>-2</sup>] <math>F\phi = (0.4810002 + 11.5331/Re_d) \times ((1-\phi^2)/\phi^2)</math> (<math>R^2 = 93.9\%</math> with a p-value = 0.000); (ii) estimating <math>F\phi</math> based on the screen thickness Reynolds number (<math>Re_t</math>) and screen porosity <math>\phi</math> [m<sup>2</sup> m<sup>-2</sup>] <math>F\phi = (0.475502 + 26.2114/Re_t) \times ((1-\phi^2)/\phi^2)</math> (<math>R^2 = 92.1\%</math> with a p-value = 0.000); (iii) estimating screen permeability <math>K_p = Dh^2\phi^3 / (2.0679(1-\phi)^2 + 3.8362 \times 10^{-10})</math> (<math>R^2 = 56.3\%</math> // 56.2 % with a p-value &lt; 0.05) as a function of thread diameter <math>D_h</math> [m] and porosity <math>\phi</math> [m<sup>2</sup> m<sup>-2</sup>]; (iv) estimating the inertial factor <math>Y = 0.0571195 + 0.135966 \cdot Dh/D_i</math> (<math>R^2 = 58.1\%</math> with a p-value = 0.0000) as a function of thread diameter [m] and the inner pore diameter <math>D_i</math> [m]. These models gave improved accuracy compared with the previous models described in the literature. Models for aerodynamic parameters of the insect-proof screens <math>K_p</math> and <math>Y</math> based in their geometric characteristics are very important to simulate the effects of insect screens in ventilation studies using computational fluid dynamic (CFD) studies.</p>
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Different models estimated aerodynamic characteristics from geometric parameters

Pressure drop coefficient estimated from screen porosity using  $Re$  based on thread diameter

Pressure drop coefficient also estimated from porosity and  $Re$  based on thread thickness

Permeability estimated from pore surface area or thread diameter & screen porosity

Inertial factor estimated from screen porosity or thread diameter & inner pore diameter

# 1 **Models for characterising the aerodynamics of insect-proof screens from their geometric** 2 **parameters**

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## 11 **Abstract**

12 This study characterised the geometric and aerodynamic parameters of 35 insect-proof screens  
 13 with different weft and warp threads, with porosities ranging from 0.237 to 0.556 m<sup>2</sup> m<sup>-2</sup>. The  
 14 geometric parameters were assessed by analysing digital images, and the aerodynamic  
 15 parameters were determined using tests in a low-speed wind tunnel. Using the experimental  
 16 measurements, four different models were developed and validated to estimate the aerodynamic  
 17 parameters of an insect-proof screen from two or more of their geometric parameters: (i) to  
 18 estimate the pressure drop coefficient  $F_\varphi$  based on the thread diameter Reynolds number ( $Re_d$ )  
 19 and screen porosity  $\varphi$  [m<sup>2</sup> m<sup>-2</sup>]  $F_\varphi = (0.4810002 + 11.5331/Re_d) \times ((1 - \varphi^2)/\varphi^2)$  ( $R^2 = 93.9\%$  with a  
 20  $p$ -value = 0.000); (ii) estimating  $F_\varphi$  based on the screen thickness Reynolds number ( $Re_t$ ) and  
 21 screen porosity  $\varphi$  [m<sup>2</sup> m<sup>-2</sup>]  $F_\varphi = (0.475502 + 26.2114/Re_t) \times ((1 - \varphi^2)/\varphi^2)$  ( $R^2 = 92.1\%$  with a  $p$ -  
 22 value = 0.000); (iii) estimating screen permeability  $K_p = D_h^2 \varphi^3 / (2.0679 (1 - \varphi)^2) + 3.8362 \times 10^{-10}$   
 23 ( $R^2 = 56.3\% // 56.2\%$  with a  $p$ -value < 0.05) as a function of thread diameter  $D_h$  [m] and porosity  
 24  $\varphi$  [m<sup>2</sup> m<sup>-2</sup>]; (iv) estimating the inertial factor  $Y = 0.0571195 + 0.135966 \cdot D_h/D_i$  ( $R^2 = 58.1\%$  with a  
 25  $p$ -value = 0.0000) as a function of thread diameter [m] and the inner pore diameter  $D_i$  [m]. These

26 models gave improved accuracy compared with the previous models described in the literature.  
27 Models for aerodynamic parameters of the insect-proof screens  $K_p$  and  $Y$  based in their  
28 geometric characteristics are very important to simulate the effects of insect screens in  
29 ventilation studies using computational fluid dynamic (CFD) studies.

30 **Keywords:** insect-proof screen; aerodynamic characterisation; aerodynamic model.

31 **Nomenclature**

32 **Abbreviations:**

33 *MD* Bias

34 *NSE* Nash-Sutcliffe efficiency

35 *PBIAS* Percent bias

36 *RMSD* Root mean squared deviation

37 *RSR* *RMSD*-observations standard deviation ratio

38 **Symbols:**

39  $a$  and  $b$  Second-order polynomial regression coefficients

40  $e$  Thickness [ $\mu\text{m}$ ]

41  $n$  Number of values

42  $u$  Air speed [ $\text{m s}^{-1}$ ]

43  $x$  Direction of airflow

44  $A$  and  $B$  Constants that depend on the type of porous material

45  $C_d$  Total discharge coefficient of a greenhouse opening

46  $C_{d,\varphi}$  Discharge coefficient due to the presence of insect-proof screens

47  $D_f$  Thread density according to the manufacturer [ $\text{threads cm}^{-2}$ ]

48  $D_h$  Diameter of the threads [ $\mu\text{m}$ ]

49  $D_{hx}$  Diameter of the weft threads [ $\mu\text{m}$ ]

50  $D_{hy}$  Diameter of the warp threads [ $\mu\text{m}$ ]

51	$D_i$	Diameter of the inside circumference of the pore [ $\mu\text{m}$ ]
52	$D_r$	Thread density measurement [threads $\text{cm}^{-2}$ ]
53	$F_\varphi$	Pressure drop coefficient due to the presence of an insect-proof screen
54	$K_p$	Screen permeability [ $\text{m}^2$ ]
55	$L_{px}$	Length of the pore in the direction of the weft [ $\mu\text{m}$ ]
56	$L_{py}$	Length of the pore in the direction of the warp [ $\mu\text{m}$ ]
57	$P$	Pressure [Pa]
58	$R$	Coefficient of correlation
59	$R^2$	Coefficient of determination
60	$Re_p$	Reynolds number based on the thread screen's permeability
61	$Re_d$	Reynolds number based on the diameter of the threads of the screen
62	$Re_t$	Reynolds number based on the thickness of the screen.
63	$S_p$	Area of the pore [ $\text{mm}^2$ ]
64	$Y$	Inertial factor

65 **Greek Symbols:**

66	$\mu$	Dynamic viscosity of air [ $\text{kg s}^{-1} \text{m}^{-1}$ ]
67	$\varphi$	Porosity [ $\text{m}^2 \text{m}^{-2}$ ]
68	$\beta$	Constant that depend on the type of porous material
69	$\rho_a$	Density of air [ $\text{kg m}^{-3}$ ]
70	$\Delta$	Difference

71 **Subscripts:**

72	<i>obs</i>	Values observed experimentally with wind tunnel tests
73	<i>sim</i>	Values simulated or obtained with models

74 **1. Introduction**

75 Insect-proof screens, placed in greenhouse vents, are used throughout the world to protect crops  
76 against insects pests. In many regions, such as the Mediterranean, screens are normally fitted  
77 to all vents in intensive greenhouse production. A recently published study showed that 99.1%  
78 of farmers in the province of Almería (Spain) install insect-proof screens in all side vents, and  
79 95.4% of farmers install them in the roof vents of their greenhouses (Valera *et al.*, 2016). The  
80 survey showed that 58.3% of the screens installed in the vents had a thread density of 15×30  
81 threads cm<sup>-2</sup> and 25.6% had a density of 10×20 threads cm<sup>-2</sup>; the percentages for roof vents  
82 were 56.0% and 22.5%, respectively (Valera *et al.*, 2016). The use of these screens reduces the  
83 number of insect that enter the greenhouse, thereby decreasing the need for crop protection  
84 chemical treatments (Berlinger *et al.*, 1993; Taylor *et al.*, 2001; Teitel, 2007) and preventing  
85 beneficial insects, such as pollinating insects (Teitel, 2007) or those used in integrated insect  
86 pest management, from leaving the greenhouse.

87 Numerous studies show the disadvantages of installing these screens, which adversely affect  
88 the ventilation rate of a greenhouse (defined as the number of times that the greenhouse air is  
89 renewed in one hour, h<sup>-1</sup>) and, therefore, its microclimate (Muñoz *et al.*, 1999; Miguel and  
90 Silva, 2000; Bartzanas *et al.*, 2002; Fatnassi *et al.*, 2002, 2003 and 2006; Soni *et al.*, 2005;  
91 Harmanto *et al.*, 2006; Kittas *et al.*, 2008; Teitel, 2007 and 2010; López *et al.*, 2014). Compared  
92 with greenhouses without insect-proof screens, greenhouses with screens have higher  
93 temperature and humidity (Bartzanas *et al.*, 2002; Fatnassi *et al.*, 2002, 2003 and 2006;  
94 Harmanto *et al.*, 2006), lower air velocity (Kittas *et al.*, 2008) and a greater vertical temperature  
95 gradient (Soni *et al.*, 2005); all these characteristics can adversely affect crop growth and  
96 development (Kittas *et al.*, 2002; Teitel, 2010).

97 Insect-proof screens affect greenhouse ventilation flow and produces an additional pressure  
98 drop to that produced by the geometry of the windows (characterised by a vent discharge  
99 coefficient  $C_d$ ). This pressured drop can be related to the velocity of air crossing the openings

100 ( $u$ ) (Teitel, 2007; Molina-Aiz *et al.*, 2009) using only a quadratic term ( $\Delta P = a u^2$ ) or a quadratic  
101 polynomial ( $\Delta P = a u^2 + b u$ ). In the first case, the aerodynamic behaviour of the screens is  
102 characterised using only one parameter, a pressure drop coefficient ( $F_\phi$ ) but in the second case,  
103 two parameters are required, permeability ( $K_p$ ), that is independent of the nature of the fluid but  
104 depends on the geometry of the porous medium, and an inertial factor ( $Y$ ) that varies with the  
105 nature of the porous medium but can be as small as 0.1 in the case of foam metal fibres (Nield  
106 and Bejan, 1999). Thus, knowing the aerodynamic characteristics of insect-proof screens ( $F_\phi$   
107 or  $K_p$  and  $Y$ ) enables the effects of their use in a greenhouse with a known ventilation rate to be  
108 estimated.

109 Accurate determination of the aerodynamic characteristics of insect-proof screens requires  
110 either wind tunnel tests and assessment of the geometric characteristics of screens. Models have  
111 been developed for estimating the pressure drop coefficient of a screen ( $F_\phi$ ) from its Reynolds  
112 number based on the thread diameter ( $Re_d$ ) and from its porosity ( $\phi$ ) (Hayama *et al.*, 2000;  
113 Pinker and Herbert, 1967; Linker *et al.*, 2002; Bailey *et al.*, 2003). Both thread diameter ( $D_h$ )  
114 and porosity ( $\phi$ ), can be determined by processing digital images of insect-proof screens  
115 (Álvarez *et al.*, 2012). Porosity can be estimated using most image processing software to count  
116 black and white pixels, or it can be estimated using specific software such as Euclides v1.4,  
117 proposed by Álvarez *et al.* (2012). This specialist software determines screen porosity by  
118 identifying the coordinates of the vertices of the screen pores, thus providing a value of screen  
119 porosity while maintaining an adequate ratio between the areas of the image occupied by pores  
120 and threads. The thread diameter of the screen can also be assessed through direct  
121 measurements with a micrometer, but manually determining thread diameter can be a very  
122 laborious task.

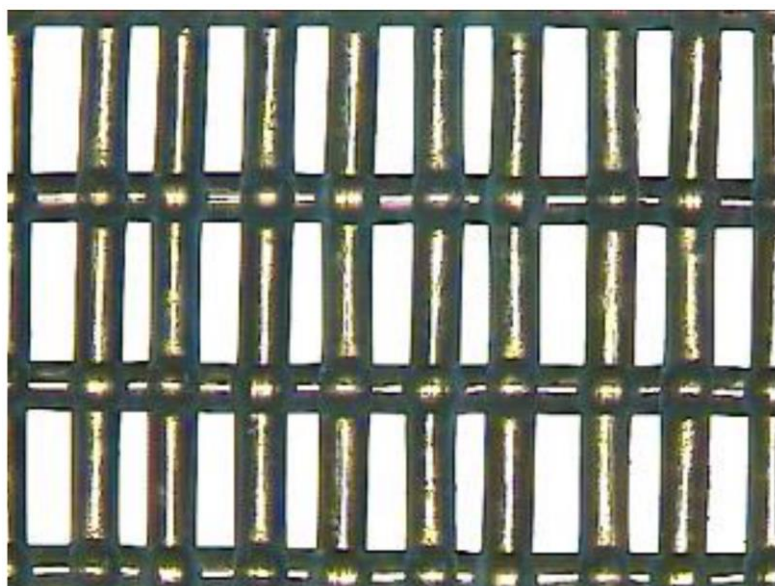
123 The objective of this study was to develop different models for estimating the pressure drop  
124 caused by an insect-proof screen at a specific air velocity from its geometric characteristics.



125 Thus, depending on the ability to estimate geometric or other parameters, a specific model may  
126 be selected to estimate the aerodynamic characteristics of an insect-proof screen. The first  
127 model investigated estimated  $F_\varphi$  as a function of porosity ( $\varphi$ ) and Reynolds number ( $Re_t$ ) based  
128 on screen thickness ( $e$ ); the second model estimated  $F_\varphi$  as a function of porosity ( $\varphi$ ) and  
129 Reynolds number  $Re_d$  based on screen thread diameter ( $D_h$ ). After estimating  $F_\varphi$ , a graph of  
130 pressure drop as a function of air velocity can be developed. A third model was proposed for  
131 estimating screen permeability ( $K_p$ ) as a function of thread diameter ( $D_h$ ) and porosity ( $\varphi$ ). A  
132 further model was proposed for estimating screen inertial factor ( $Y$ ) as a function of thread  
133 diameter ( $D_h$ ) and inner pore diameter ( $D_i$ ). After estimating  $K_p$  and  $Y$ , and knowing the screen  
134 thickness ( $e$ ), a screen pressure drop graph can be developed.

## 135 **2. Material and methods**

136 To develop different models for estimating the aerodynamic characteristics of an insect-proof  
137 screen from only its geometric characteristics, 35 insect-proof screens with different thread  
138 densities were analysed (Table 1). Insect-proof screens are manufactured with high-density  
139 polyethylene (HDPE) monofilament-woven fabrics, with knot-free weft and warp threads (Fig.  
140 1). This type of insect-proof screen is used in all greenhouses in the province of Almería, Spain  
141 (Valera *et al.* 2016) and is the type most commonly used in greenhouses worldwide.



142

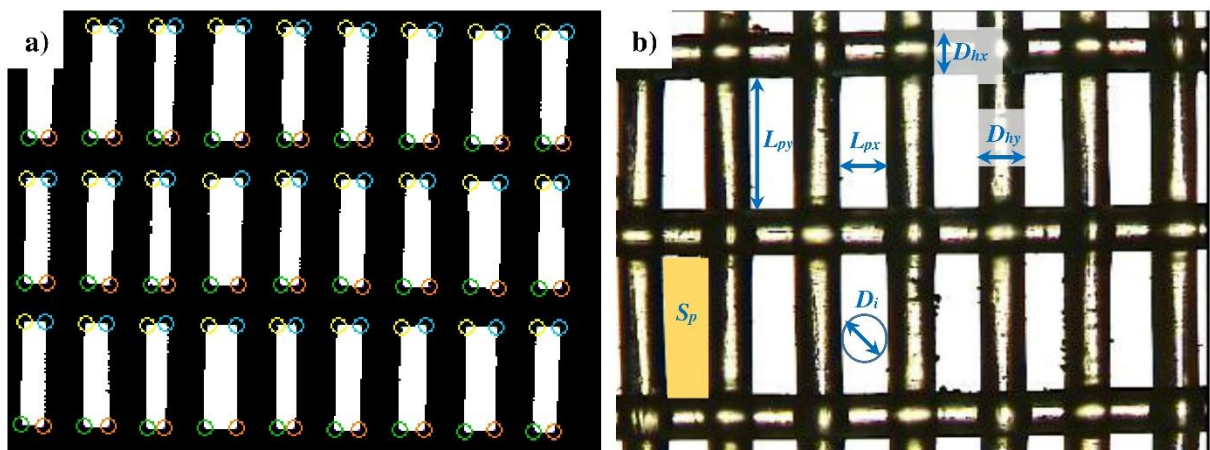
143 **Fig. 1.** Microscopic image (4×) of a 14×27 threads cm<sup>-2</sup> insect-proof screen

144

### 145 **2.1. Determining the geometric characteristics of insect-proof screens**

146 The two-dimensional geometric characteristics of insect-proof screens were assessed by  
147 processing digital images acquired with a Motic DMWB1-223 microscope (Motic Spain S.L.,  
148 Barcelona) equipped with a digital camera, and with a 4× calibrated microscope lens (with a  
149 resolution of 10.5 μm pixel<sup>-1</sup>). Three samples were analysed for each screen, acquiring 24  
150 × 0.25 cm<sup>2</sup> images per sample. The method used to determine the geometric characteristics of  
151 insect-proof screens is available from López *et al.* (2013). The specific software Euclides v1.4  
152 (Álvarez, 2010; Álvarez *et al.*, 2012) was used to identify the vertices of each pore (Fig. 2a).  
153 From these vertices, the following two-dimensional geometric parameters were obtained (Table  
154 1 and Fig. 2b):  $D_r$ , measured thread density [threads cm<sup>-2</sup>];  $\phi$ , porosity [m<sup>2</sup> m<sup>-2</sup>];  $L_{px}$  and  $L_{py}$ ,  
155 the lengths of the pore [μm] in the weft and warp directions, respectively;  $D_{hx}$  and  $D_{hy}$ , diameter  
156 [μm] of the weft and warp threads, respectively;  $D_h$ , diameter of the threads [μm];  $D_i$ , inner  
157 pore diameter [μm];  $S_p$ , pore area [mm<sup>2</sup>]. For each image acquired, approximately 12 (for 10 ×  
158 20 threads cm<sup>-2</sup> screens), 27 (for 14 × 27 threads cm<sup>-2</sup> screens) and 30 (for 13 × 30 threads cm<sup>-2</sup>  
159 screens) pores were analysed giving a total of 864, 1944 and 2160 pores analysed per screen,  
160 respectively. Álvarez *et al.* (2012) made an exhaustive analysis where they showed that three  
161 randomly selected samples were sufficient to geometrically characterise these types of screen.

162



163 **Fig. 2.** Image of a  $14 \times 27$  threads  $\text{cm}^{-2}$  insect-proof screen with identified pore vertices (a).

164 Geometric parameters indicated on a  $10 \times 20$  threads  $\text{cm}^{-2}$  screen (b).

165

166 The thickness ( $e$ ) of the insect-proof screens has been measured with a TESA-VISIO 300 device

167 (TESA SA, Switzerland; resolution of  $0.05 \mu\text{m}$ ) through non-contact optical measurement. For

168 the magnitude of the measurements  $e$ , the uncertainty was  $<10 \mu\text{m}$ .

169

170 **Table 1.** Geometric characteristics of the insect-proof screens ( $D_f$ , thread density according to  
171 the manufacturer [threads  $\text{cm}^{-2}$ ]). Average value and standard deviation of:  $D_r$ , measured  
172 thread density [threads  $\text{cm}^{-2}$ ];  $\varphi$ , porosity [ $\text{m}^2 \text{m}^{-2}$ ];  $L_{px}$  and  $L_{py}$ , the lengths of the pore [ $\mu\text{m}$ ] in  
173 the weft and warp directions, respectively;  $D_{hx}$  and  $D_{hy}$ , diameter [ $\mu\text{m}$ ] of the weft and warp  
174 threads, respectively;  $D_h$ , diameter of the threads [ $\mu\text{m}$ ];  $D_i$ , diameter of the inside  
175 circumference of the pore [ $\mu\text{m}$ ];  $S_p$ , area of the pore [ $\text{mm}^2$ ];  $e$ , thickness [m].

$N$	$D_f$	$D_r$	$\varphi$	$L_{px}$	$L_{py}$	$D_{hx}$	$D_{hy}$	$D_h$	$D_i$	$S_p$	$e$ [ $\times 10^{-6}$ ]
1	11×23	10.4×22.2	0.322±0.008	197.0±15.1	709.5±30.5	248.2±8.4	254.4±10.6	251.3±10.3	202.0±15.1	0.140±0.012	589±7
2*	10×20	9.8×20.1	0.369±0.012	243.7±25.1	774.0±61.0	251.6±7.7	253.5±8.3	252.5±8.1	248.5±25.0	0.189±0.025	596±8
3	10×20	10.1×20.6	0.366±0.006	232.5±18.8	760.7±25.7	233.1±6.2	253.1±9.0	243.1±12.6	237.4±18.6	0.177±0.015	544±7
4	10×20	10.7×21.3	0.349±0.009	226.9±20.1	681.1±26.5	256.8±6.0	243.5±11.3	250.2±11.6	232.1±20.3	0.154±0.015	567±5
5	10×20	9.7×20.4	0.402±0.008	252.9±23.4	806.9±28.2	223.4±7.4	238.5±9.0	230.9±11.1	257.5±23.2	0.204±0.020	501±7
6	10×20	10.3×21.5	0.310±0.008	199.2±15.6	710.8±30.7	264.4±9.4	267.1±11.7	265.7±11.0	203.7±15.7	0.141±0.013	571±19
7	10×20	10.4×21.0	0.302±0.010	199.9±28.8	693.6±20.7	267.3±7.1	276.5±15.0	271.9±13.4	204.3±28.9	0.139±0.020	536±6
8	10×20	9.0×20.7	0.402±0.010	246.8±16.4	877.3±16.6	233.8±6.3	236.5±9.2	235.1±8.4	249.8±16.6	0.216±0.015	546±4
9	14×27	13.7×26.9	0.379±0.014	187.3±34.8	543.5±26.1	186.5±7.0	184.0±8.4	185.2±7.9	192.5±34.6	0.102±0.020	418±8
10	10×20	9.7×19.7	0.378±0.009	253.9±17.9	784.3±54.2	250.5±7.9	253.5±8.2	252.0±8.2	255.8±17.9	0.199±0.020	587±7
11	10×20	8.9×19.5	0.375±0.007	251.7±23.9	863.6±25.0	264.4±7.1	260.7±11.5	262.5±10.2	256.0±24.1	0.217±0.022	604±9
12	10×20	8.9×19.6	0.375±0.007	250.3±23.9	865.1±27.1	264.6±7.7	260.3±11.7	262.4±10.6	255.8±24.9	0.216±0.022	611±18
13	10×20	9.9×19.7	0.368±0.007	252.7±21.2	746.4±34.9	259.0±7.0	255.7±9.4	257.3±8.7	255.2±21.0	0.189±0.018	639±7
14	14×27	12.0×27.5	0.292±0.009	141.8±28.9	615.9±15.9	214.8±7.2	221.7±8.5	218.3±8.7	144.2±28.8	0.087±0.018	514±5
15	14×27	12.8×27.9	0.267±0.007	131.8±26.5	570.5±15.2	209.6±6.9	225.7±8.2	217.7±11.1	134.1±26.7	0.075±0.015	490±10
16	10×20	9.9×21.8	0.338±0.017	207.4±24.1	756.6±28.3	253.4±10.0	251.8±12.4	252.6±11.6	210.7±24.5	0.156±0.016	540±10
17	15×30	15.2×30.2	0.556±0.030	221.6±19.6	548.8±8.7	110.5±4.2	109.9±5.1	110.1±4.8	222.9±19.5	0.121±0.011	261±6
18	18×31	18.6×31.3	0.520±0.023	209.0±12.0	427.7±7.7	110.6±4.3	110.2±4.3	110.4±4.3	210.3±11.9	0.089±0.006	259±14
19	16×30	16.1×30.8	0.368±0.026	162.2±10.6	458.4±17.8	163.1±5.3	162.8±6.3	162.9±5.9	164.0±10.7	0.070±0.006	362±4
20	14×30	14.3×30.7	0.385±0.026	162.6±10.8	540.6±17.5	159.8±5.7	163.5±6.5	161.9±6.4	165.0±10.8	0.088±0.007	371±6
21	12×30	12.1×30.4	0.405±0.027	166.7±11.7	663.6±25.0	164.7±5.8	162.9±5.8	163.6±5.8	169.6±12.0	0.111±0.0009	388±6
22	10×30	9.6×30.2	0.437±0.029	170.9±13.8	876.8±27.9	163.3±5.3	160.0±5.9	161.2±5.9	174.5±14.2	0.150±0.013	406±6
23	10×30	9.4×30.2	0.446±0.043	173.0±20.0	900.9±125.4	158.1±5.6	158.4±5.4	158.3±5.5	177.2±20.0	0.156±0.029	419±27
24	13×30	13.1×30.5	0.390±0.006	164.6±9.3	593.3±19.0	168.6±6.6	163.1±6.3	165.5±7.0	167.4±9.6	0.098±0.006	392±5
25	10×20	9.9×19.7	0.335±0.011	233.7±23.9	734.0±29.2	276.4±11.2	273.4±10.7	274.5±11.0	236.6±24.0	0.171±0.019	564±6
26	14×27	12.9×26.8	0.385±0.006	188.4±25.7	591.6±28.2	184.1±7.2	184.7±7.1	184.4±7.1	191.3±25.6	0.111±0.016	402±15
27	10×20	9.2×20.7	0.375±0.007	234.9±16.1	838.7±27.0	245.8±7.1	248.0±8.3	247.2±7.9	238.7±16.4	0.197±0.015	526±26
28	10×20	10.1×20.0	0.379±0.007	256.6±14.3	736.4±17.1	256.8±8.3	243.7±8.2	248.6±10.4	259.8±14.4	0.189±0.011	480±11
29	13×30	12.5×31.3	0.263±0.005	110.0±7.9	611.9±17.5	187.7±6.7	209.4±8.2	200.2±13.1	113.5±8.5	0.067±0.005	458±44
30	10×20	9.8×20.0	0.350±0.008	238.6±19.5	746.0±22.7	272.0±7.2	261.2±12.4	265.3±11.9	241.7±19.5	0.178±0.015	564±45

31	15×30	15.3×31.4	0.237±0.006	107.5±14.6	456.3±20.8	196.0±7.6	211.1±7.6	204.7±10.7	110.7±16.7	0.049±0.008	508±33
32	10×20	9.6×20.3	0.360±0.007	239.9±18.5	765.4±27.1	272.2±11.6	252.0±8.6	259.6±13.9	241.9±19.1	0.182±0.015	508±19
33	10×20	9.2×19.9	0.381±0.008	250.6±17.7	829.1±37.1	257.5±8.5	251.3±8.6	253.6±9.1	255.0±17.7	0.208±0.017	504±12
34	10×20	9.7×19.9	0.381±0.014	254.1±22.3	777.7±25.3	253.6±6.9	247.2±11.1	249.6±10.2	257.2±22.6	0.198±0.018	535±7
35	10×20	9.8×19.9	0.354±0.007	240.0±18.0	761.5±23.7	264.0±7.8	261.8±9.3	262.6±8.8	242.8±17.8	0.183±0.015	535±10

176 \* Screens 2 to 22 were analysed by Álvarez (2010).

## 177 **2.2. Determining the aerodynamic characteristics of insect-proof screens**

178 The aerodynamic characteristics of insect-proof screens were determined by tests using a wind  
179 tunnel (Fig. 3) designed at the University of Almería, Spain (Molina-Aiz *et al.*, 2006; Valera *et*  
180 *al.*, 2006), with an improved control system (Espinoza *et al.*, 2015). The wind tunnel has a total  
181 length of 4.74 m and a 388 mm diameter test section, where the insect-proof screen samples  
182 were placed. Three tests were performed for each screen, with different, randomly selected  
183 samples (Espinoza *et al.*, 2015; López *et al.*, 2018). The wind tunnel had a maximum speed of  
184 10 m s<sup>-1</sup> and a maximum pressure drop of 200 Pa. These limits are set based on the air velocity  
185 characteristics and the wind tunnel pressure drop sensors. The pressure drop was recorded by  
186 two Pitot tubes (Airflow Developments Ltd, Buckinghamshire, UK; 4 mm of diameter) and  
187 with a differential pressure transducer SI727 (Special Instruments, Norlingen, Germany;  
188 operational range of 0-200 Pa and accuracy of ±0.25%). Air velocity and air temperature were  
189 measured by a hot-wire anemometer EE70-VT32C5 (Elektronik, Engerwitzdorf, Austria;  
190 measurement range of 0-10 m s<sup>-1</sup> and 0-50 °C; accuracy of ±0.2 m s<sup>-1</sup> + 2 % of measuring value  
191 and ±0.2 °C). The technical characteristics of the wind tunnel and the control system used in  
192 this study are described in Espinoza *et al.* (2015). For each sample, a test sequence with nine  
193 increasing airspeeds or pressure drops was performed, followed by a sequence with nine  
194 decreasing airspeeds or pressure drops.



**Fig. 3.** Wind tunnel used for the tests

195

196

197

198 Curves of pressure drop  $\Delta P$  [Pa] as a function of air velocity  $u$  [ $\text{m s}^{-1}$ ] were constructed for each  
 199 insect-proof screen. Insect-proof screens are porous media from which their aerodynamic  
 200 parameters can be determined:  $K_p$ , the screen permeability [ $\text{m}^2$ ], is a coefficient that depends  
 201 on the geometry of the porous medium, and is independent of the nature of the fluid (Nield and  
 202 Bejan, 1999);  $Y$ , the inertial factor [dimensionless], a drag constant that depends on the  
 203 characteristics of the porous material;  $F_\phi$ , the pressure drop coefficient.

204 Airflow through a porous medium can be described by modifying Darcy's equation  
 205 (Forchheimer, 1901):

$$206 \quad \frac{\partial P}{\partial x} = - \left( \frac{\mu}{K_p} u + \rho_a \left( \frac{Y}{K_p^{1/2}} \right) |u| u \right) \quad (1)$$

207 where  $P$  is the pressure [Pa],  $x$  is the airflow direction,  $u$  is the air velocity [ $\text{m s}^{-1}$ ],  $\mu$  is the  
 208 dynamic viscosity of air [ $\text{kg s}^{-1} \text{m}^{-1}$ ] and  $\rho_a$  is the air density [ $\text{kg m}^{-3}$ ]. To calculate both  $\mu$  and  
 209  $\rho_a$  it was necessary to measure the air temperature (Molina-Aiz *et al.*, 2004). Air temperature  
 210 remained practically constant in each test (ranging from 19.8 and 25.5 °C). Some authors have  
 211 used a second-degree polynomial to fit the experimentally observed pressure drop values as a  
 212 function of the airspeed passing through the porous medium (Miguel *et al.*, 1997; Dierickx,  
 213 1998; Muñoz *et al.*, 1999; Molina-Aiz *et al.*, 2006; Valera *et al.*, 2006). The zero-order term

214 can be neglected (Miguel *et al.*, 1997; Molina-Aiz *et al.*, 2009; López *et al.*, 2014), leading to  
 215 the following equation:

$$216 \quad \Delta P = au^2 + bu \quad (2)$$

217 By matching the first and second order coefficients of Eq. (2) with Eq. (1), the screen  
 218 permeability ( $K_p$ ) and its inertial factor ( $Y$ ) can be determined as follows (Molina-Aiz *et al.*,  
 219 2009):

$$220 \quad K_p = e \frac{\mu}{b} \quad (3)$$

$$221 \quad Y = \frac{a K_p^{0.5}}{\rho_a e} \quad (4)$$

222 Applying Eqs. (3) and (4) requires knowledge of the thickness ( $e$ ) of the screens. Another way  
 223 of describing the relationship between the pressure drop and airspeed is to use Bernoulli's  
 224 equation (Kosmos *et al.*, 1993; Montero *et al.*, 1997; Teitel and Shklyar, 1998):

$$225 \quad \Delta P = -1/2 F_\phi \rho_a u^2 \quad (5)$$

226 where  $F_\phi$  is the pressure drop coefficient due to the presence of an insect-proof screen. The  
 227 coefficient  $F_\phi$  can be determined using Eqs. (1) and (5) since  $\partial P/\partial x = \Delta P/e$  (Molina-Aiz *et al.*,  
 228 2009):

$$229 \quad F_\phi = \frac{2e}{K_p^{0.5}} \left( \frac{1}{Re_p} + Y \right) \quad (6)$$

230 Below a specific Reynolds number limit, Teitel (2001) showed that the coefficient  $F_\phi$  can be  
 231 used to predict the pressure drop due to the screen. For a permeability-based Reynolds number  
 232 ( $Re_p$ ), the limit can be set to  $Re_p < 10^5$  (Molina-Aiz *et al.*, 2009). The permeability Reynolds  
 233 number ( $Re_p$ ) can be calculated by (Nield and Bejan, 1999):

$$234 \quad Re_p = \frac{\sqrt{K_p} u \rho_a}{\mu} \quad (7)$$

235 Some authors have presented statistical models for predicting the value of  $F_\phi$  as a function of  
 236 porosity  $\phi$  [ $\text{m}^2 \text{m}^{-2}$ ] and Reynolds number ( $Re_d$ ) based on the thread diameter ( $D_h$ ) [m]: Eq. (8)

237 presented by Hayama *et al.* (2000) (in Bailey *et al.*, 2003), Eq. (9) by Pinker and Herbert (1967)  
 238 and Eq. (10) by Bailey *et al.* (2003):

$$239 \quad F_\varphi = 28 \left( \frac{Re_d \varphi^2}{1-\varphi} \right)^{-0.95} \quad (8)$$

$$240 \quad F_\varphi = \left( \frac{13.0}{Re_d} + 0.82 \right) \left( \frac{1-\varphi^2}{\varphi^2} \right) \quad (9)$$

$$241 \quad F_\varphi = \left[ \frac{18}{Re_d} + \frac{0.75}{\log(Re_d+1.25)} + 0.055 \log(Re_d) \right] \left[ \frac{1-\varphi^2}{\varphi^2} \right] \quad (10)$$

242 The Reynolds number  $Re_d$  is calculated from the air velocity [m s<sup>-1</sup>] and the thread diameter  
 243 ( $D_h$ ) [m]:

$$244 \quad Re_d = \frac{D_h u \rho_a}{\mu} \quad (11)$$

245 Another way of predicting the pressure drop in porous media is to use the equations presented  
 246 in Nield and Bejan (1999) to estimate the  $K_p$  and  $Y$  values. The permeability ( $K_p$ ) of the screen  
 247 can be estimated from its porosity  $\varphi$  [m<sup>2</sup> m<sup>-2</sup>] and fibre thread diameter  $D_h$  [m]. This expression  
 248 is known as the Kozeny equation (Nield and Bejan, 1999):

$$249 \quad K_p = \frac{D_h^2 \varphi^3}{\beta(1-\varphi)^2} \quad (12)$$

250 where  $\beta$  is a constant that depends on the type of porous material. The inertial factor  $Y$  can be  
 251 estimated from the fibre diameter and pore size (Beavers *et al.*, 1973, Nield and Bejan, 1999),  
 252 according to the following linear fit:

$$253 \quad Y = A + B \frac{D_h}{D_i} \quad (13)$$

254 in which  $A$  and  $B$  are two constants that depend on the type of porous material. In the case of  
 255 insect-proof screens, we should use the thread diameter  $D_h$  and the inner pore diameter  $D_i$ .

256 Lastly, another method to estimate  $K_p$  and  $Y$  is to use the equations proposed by Miguel (1998)  
 257 from the porosity  $\varphi$  [m<sup>2</sup> m<sup>-2</sup>]:

$$258 \quad K_p = 3.44 \times 10^{-9} \varphi^{1.6} \quad (14)$$

$$259 \quad Y = 4.30 \times 10^{-2} \varphi^{-2.13} \quad (15)$$



260 After determining the aerodynamic characteristics of the 35 insect-proof screens in the wind  
 261 tunnel, the goodness of fit of the proposed models was analysed by comparing the  
 262 experimentally observed values of  $F_\phi$  (tunnel tests) with the corresponding estimated values of  
 263  $F_\phi$  according to Eqs. (8), (9) and (10), and proposing alternative models for estimating  $F_\phi$ . The  
 264 goodness of fit of the proposed models was also analysed by comparing the experimentally  
 265 observed results (tunnel tests) and estimated values of  $K_p$  and  $Y$  according to Eqs. (12) and (13)  
 266 and Eqs. (14-15), and similarly proposing alternative models for estimating  $K_p$  and  $Y$ .

### 267 **2.3. Statistical analysis**

268 Regression analyses were used to compare the different variables to obtain statistically  
 269 significant relationships ( $p$ -value  $< 0.05$ ) using Statgraphics® Centurion 18 v18.1 (Statgraphics  
 270 Technologies, Inc., The Plains, VA, USA). In addition to the correlation coefficient,  $R$ , other  
 271 statistics were used to analyse the fit of the simulated (using different models) values of  $F_\phi$ ,  $K_p$   
 272 and  $Y$  to the experimentally observed values. Two of the most commonly used statistics for  
 273 goodness of fit, based on the deviation of  $X_{sim}$  (simulated values) from  $X_{obs}$  (observed values)  
 274 are the *RMSD* (root mean squared deviation) and the *MD* (or *bias*) (Kobayashi and Salam,  
 275 2000):

$$276 \quad RMSD = \sqrt{\frac{1}{n} \sum_{i=1}^n (X_{obs} - X_{sim})^2} \quad (16)$$

$$277 \quad MD = \frac{1}{n} \sum_{i=1}^n (X_{obs} - X_{sim}) \quad (17)$$

278 The *RMSD* represents the average distance between the simulated and observed values. The  
 279 *MD* corresponds to the mean value of the differences between the simulated and observed  
 280 values, although in this case negative differences are compensated by positive ones, which may  
 281 lead to an erroneous interpretation. Both statistics represent different aspects of the deviation  
 282 from simulated values, but the relationship between is not been well defined (Kobayashi and  
 283 Salam, 2000).

284 The Nash-Sutcliffe efficiency (*NSE*) is a normalised statistic indicating the relative magnitude  
 285 of the residual variance of the model (“noise”) with respect to the variance of the observed  
 286 values (“information”) (Nash and Sutcliffe, 1970). The *NSE* indicates how well a plot of  
 287 observed versus simulated values fits the 1:1 line (Moriase *et al.*, 2007):

$$288 \quad NSE = 1 - \left[ \frac{\sum_{i=1}^n (X_{obs} - X_{sim})^2}{\sum_{i=1}^n (X_{obs} - \bar{X}_{obs})^2} \right] \quad (18)$$

289 where  $\bar{X}_{obs}$  is the average value of all  $X_{obs}$  values. The *NSE* can take values between  $-\infty$  and  
 290 1.0, the latter being the optimal value. Values between 0.0 and 1.0 are generally viewed as  
 291 acceptable levels of performance (Moriase *et al.*, 2007).

292 Percent bias (*PBIAS*) represents the mean trend of simulated values to greater or lower than  
 293 their respective observed values (Gupta *et al.*, 1999). The optimal value of *PBIAS* is 0.0, and  
 294 values close to 0 indicate high model precision. Positive values indicate that the model provides  
 295 values that are lower than those observed (model underestimation bias), whilst negative values  
 296 indicate the opposite (model overestimation bias) (Gupta *et al.*, 1999). *PBIAS* can be expressed  
 297 as a percentage:

$$298 \quad PBIAS = \left[ \frac{\sum_{i=1}^n (X_{obs} - X_{sim}) \times 100}{\sum_{i=1}^n (X_{obs})} \right] \quad (19)$$

299 *RMSD*-observations standard deviation ratio (*RSR*): this statistic standardises the values of  
 300 *RMSD* with the standard deviation from the observed values (Moriiasi *et al.*, 2007). It combines  
 301 both an error index and the additional information recommended by Legates and McCabe  
 302 (1999):

$$303 \quad RSR = \frac{RMSD}{STDEV_{obs}} = \left[ \frac{\sqrt{\sum_{i=1}^n (X_{obs} - X_{sim})^2}}{\sqrt{\sum_{i=1}^n (X_{obs} - \bar{X}_{obs})^2}} \right] \quad (20)$$

304 *RSR* takes values from 0 to a large positive value, and when *RMSD* is equal to zero, the model  
 305 is considered perfect. Small values of *RSR* indicate better performance of the simulation model  
 306 (Moriiasi *et al.*, 2007).

307 **3. Results and discussion**

308 Here, the results from the wind tunnel tests of the 35 insect-proof screens are presented. Two  
 309 different models for estimating  $F_\phi$  values as a function of different geometric screen  
 310 characteristics are then presented, comparing the performance of these models with other  
 311 models previously described in the literature. One model for estimating  $K_p$  values and another  
 312 model for estimating  $Y$  values are presented and the performance of these models is compared  
 313 with other models previously described in the literature. The effect of porosity on pressure drop  
 314 predicted by the different models was analysed.

315 **3.1. Aerodynamic characteristics assessed in wind tunnel tests**

316 Table 2 presents the values of the aerodynamic parameters ( $K_p$ ,  $Y$  and  $F_\phi$ ) for the 35 insect-  
 317 proof screens tested in a wind tunnel.

318

319 **Table 2.** Aerodynamic characteristics of the insect-proof screens:  $a$  and  $b$  are the coefficients  
 320 of the polynomial fit from the wind tunnel tests (Eq. (2));  $R^2$ , the fit determination coefficient;  
 321  $K_p$ , screen permeability [ $m^2$ ];  $Y$ , inertial factor;  $F_\phi$ , pressure drop coefficient due to the presence  
 322 of an insect-proof screen expressed as function of  $Re_p$ .

323  $N$ : number of the screen.

$N$	$a$	$b$	$R^2$	$K_p$	$Y$	$F_\phi$
1	2.784	3.045	0.9997	$3.512 \times 10^{-9}$	0.233	$19.88 \times (Re_p^{-1} + 0.233)$
2	2.102	2.676	0.9928	$4.041 \times 10^{-9}$	0.187	$18.74 \times (Re_p^{-1} + 0.187)$
3	2.036	2.253	0.9983	$4.386 \times 10^{-9}$	0.206	$16.44 \times (Re_p^{-1} + 0.206)$
4	2.407	2.876	0.9988	$3.576 \times 10^{-9}$	0.212	$18.95 \times (Re_p^{-1} + 0.212)$
5	1.767	1.676	0.9993	$5.432 \times 10^{-9}$	0.216	$13.61 \times (Re_p^{-1} + 0.216)$
6	3.029	3.792	0.9976	$2.731 \times 10^{-9}$	0.231	$21.84 \times (Re_p^{-1} + 0.231)$
7	2.985	3.235	0.9993	$3.007 \times 10^{-9}$	0.254	$19.55 \times (Re_p^{-1} + 0.254)$
8	1.519	1.534	0.9997	$6.461 \times 10^{-9}$	0.186	$13.58 \times (Re_p^{-1} + 0.186)$
9	1.879	3.074	0.9990	$2.467 \times 10^{-9}$	0.186	$16.82 \times (Re_p^{-1} + 0.186)$
10	1.909	2.375	0.9987	$4.484 \times 10^{-9}$	0.181	$17.52 \times (Re_p^{-1} + 0.181)$
11	1.845	1.876	0.9995	$5.849 \times 10^{-9}$	0.194	$15.81 \times (Re_p^{-1} + 0.194)$
12	1.852	1.899	0.9990	$5.835 \times 10^{-9}$	0.193	$15.99 \times (Re_p^{-1} + 0.193)$

13	1.947	2.176	0.9986	$5.331 \times 10^{-9}$	0.185	$17.51 \times (\text{Re}_p^{-1} + 0.185)$
14	3.182	5.440	0.9992	$1.716 \times 10^{-9}$	0.213	$24.83 \times (\text{Re}_p^{-1} + 0.213)$
15	3.595	6.006	0.9997	$1.480 \times 10^{-9}$	0.235	$25.46 \times (\text{Re}_p^{-1} + 0.235)$
16	2.284	3.051	0.9994	$3.211 \times 10^{-9}$	0.200	$19.05 \times (\text{Re}_p^{-1} + 0.200)$
17	0.870	2.635	0.9993	$1.793 \times 10^{-9}$	0.118	$12.31 \times (\text{Re}_p^{-1} + 0.118)$
18	0.824	2.744	0.9992	$1.713 \times 10^{-9}$	0.110	$12.51 \times (\text{Re}_p^{-1} + 0.110)$
19	2.235	4.738	0.9996	$1.386 \times 10^{-9}$	0.192	$19.43 \times (\text{Re}_p^{-1} + 0.192)$
20	1.896	3.749	0.9997	$1.798 \times 10^{-9}$	0.180	$17.52 \times (\text{Re}_p^{-1} + 0.180)$
21	1.575	3.179	0.9992	$2.216 \times 10^{-9}$	0.159	$16.50 \times (\text{Re}_p^{-1} + 0.159)$
22	1.302	2.683	0.9993	$2.745 \times 10^{-9}$	0.140	$15.50 \times (\text{Re}_p^{-1} + 0.140)$
23	1.261	2.645	0.9949	$2.876 \times 10^{-9}$	0.134	$15.64 \times (\text{Re}_p^{-1} + 0.134)$
24	1.669	3.885	0.9995	$1.853 \times 10^{-9}$	0.155	$18.20 \times (\text{Re}_p^{-1} + 0.155)$
25	2.562	4.159	0.9992	$2.453 \times 10^{-9}$	0.187	$22.77 \times (\text{Re}_p^{-1} + 0.187)$
26	2.038	3.585	0.9994	$2.028 \times 10^{-9}$	0.190	$17.84 \times (\text{Re}_p^{-1} + 0.190)$
27	1.913	1.844	0.9968	$5.183 \times 10^{-9}$	0.218	$14.61 \times (\text{Re}_p^{-1} + 0.218)$
28	2.065	2.233	0.9949	$3.926 \times 10^{-9}$	0.226	$15.33 \times (\text{Re}_p^{-1} + 0.226)$
29	3.886	5.802	0.9992	$1.439 \times 10^{-9}$	0.269	$24.15 \times (\text{Re}_p^{-1} + 0.269)$
30	2.452	3.262	0.9992	$3.154 \times 10^{-9}$	0.204	$20.10 \times (\text{Re}_p^{-1} + 0.204)$
31	5.872	6.111	0.9995	$1.514 \times 10^{-9}$	0.377	$26.09 \times (\text{Re}_p^{-1} + 0.377)$
32	2.088	1.638	0.998	$5.612 \times 10^{-9}$	0.255	$13.57 \times (\text{Re}_p^{-1} + 0.255)$
33	1.915	1.310	0.999	$7.004 \times 10^{-9}$	0.266	$12.04 \times (\text{Re}_p^{-1} + 0.266)$
34	2.089	1.506	0.999	$6.427 \times 10^{-9}$	0.260	$13.35 \times (\text{Re}_p^{-1} + 0.260)$
35	2.163	2.861	0.998	$3.417 \times 10^{-9}$	0.199	$18.29 \times (\text{Re}_p^{-1} + 0.199)$

324

325 **3.2. Models for estimating  $F_\phi$  values of an insect-proof screen with known geometric**  
326 **characteristics**

327 To study greenhouse microclimate using computational fluid dynamic (CFD) simulations or  
328 other types of mathematical simulation models it is necessary to know the aerodynamic  
329 characteristics of the screens. Typically, insect-proof screen manufacturers do not provide the  
330 aerodynamic characteristics or full geometric characteristics. Therefore, researchers often resort  
331 to models such as those presented in Eqs. (8), (9), (10), (14) and (15) to estimate the  
332 aerodynamic characteristics of the screens under study. For the screens analysed in this study,  
333 alternative models were developed for estimating the values of  $F_\phi$ ,  $K_p$  and  $Y$  from different  
334 geometric parameters.

335 From the  $F_\phi$  expression determined using Eq. (6) (Table 2),  $F_\phi$  values were calculated for each  
 336 screen, at different values of airspeed  $u$  varying from 0.25 to 3.00 m s<sup>-1</sup>, in +0.25 m s<sup>-1</sup> intervals.  
 337 The maximum limit was set to 3 m s<sup>-1</sup> because this air velocity is unlikely to be reached in an  
 338 insect-proof screen in a commercial greenhouse under natural ventilation conditions. The  
 339 maximum values of air velocity observed near the vents of a commercial greenhouse with  
 340 natural ventilation (side and roof vents) are normally unlikely to exceed 1.5 m s<sup>-1</sup> (Molina-Aiz  
 341 et al., 2009; López *et al.*, 2012).

342 The first model, for estimating  $F_\phi$  given the insect-proof screen porosity  $\phi$  [m<sup>2</sup> m<sup>-2</sup>] and using  
 343 a Reynolds number  $Re_d$  based on the thread diameter  $D_h$  [m], is presented in the following  
 344 equation derived from the experimental observations (Fig. 4b):

$$345 \quad F_\phi = \left(0.481002 + \frac{11.5331}{Re_d}\right) \left(\frac{1-\phi^2}{\phi^2}\right) \quad (21)$$

346 with a 0.97 correlation coefficient, with a 93.9% R<sup>2</sup> and with a  $p$ -value<0.001.

347 This equation is similar to those reported elsewhere in the literature where the value of  $F_\phi$  is  
 348 estimated from a thread diameter Reynolds number (similar to Eq. (11)).

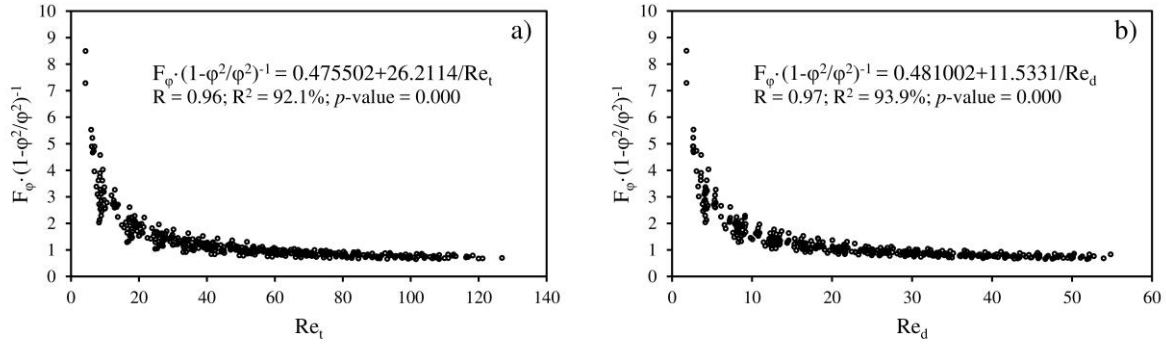
349 The second model developed is useful for estimating  $F_\phi$  when the thread diameter is difficult  
 350 to assess. From the experimental data (Fig. 4a), the equation was derived for estimating  $F_\phi$  from  
 351 porosity  $\phi$  [m<sup>2</sup> m<sup>-2</sup>] and Reynolds number  $Re_t$  based on the screen thickness  $e$  [m] which is  
 352 easier to measure with a micrometre:

$$353 \quad F_\phi = \left(0.475502 + \frac{26.2114}{Re_t}\right) \left(\frac{1-\phi^2}{\phi^2}\right) \quad (22)$$

354 This equation had a 0.96 correlation coefficient, with a 92.1% R<sup>2</sup> and with a  $p$ -value<0.001.

355 The Reynolds based on screen thickness [m] is:

$$356 \quad Re_t = \frac{eu\rho a}{\mu} \quad (23)$$

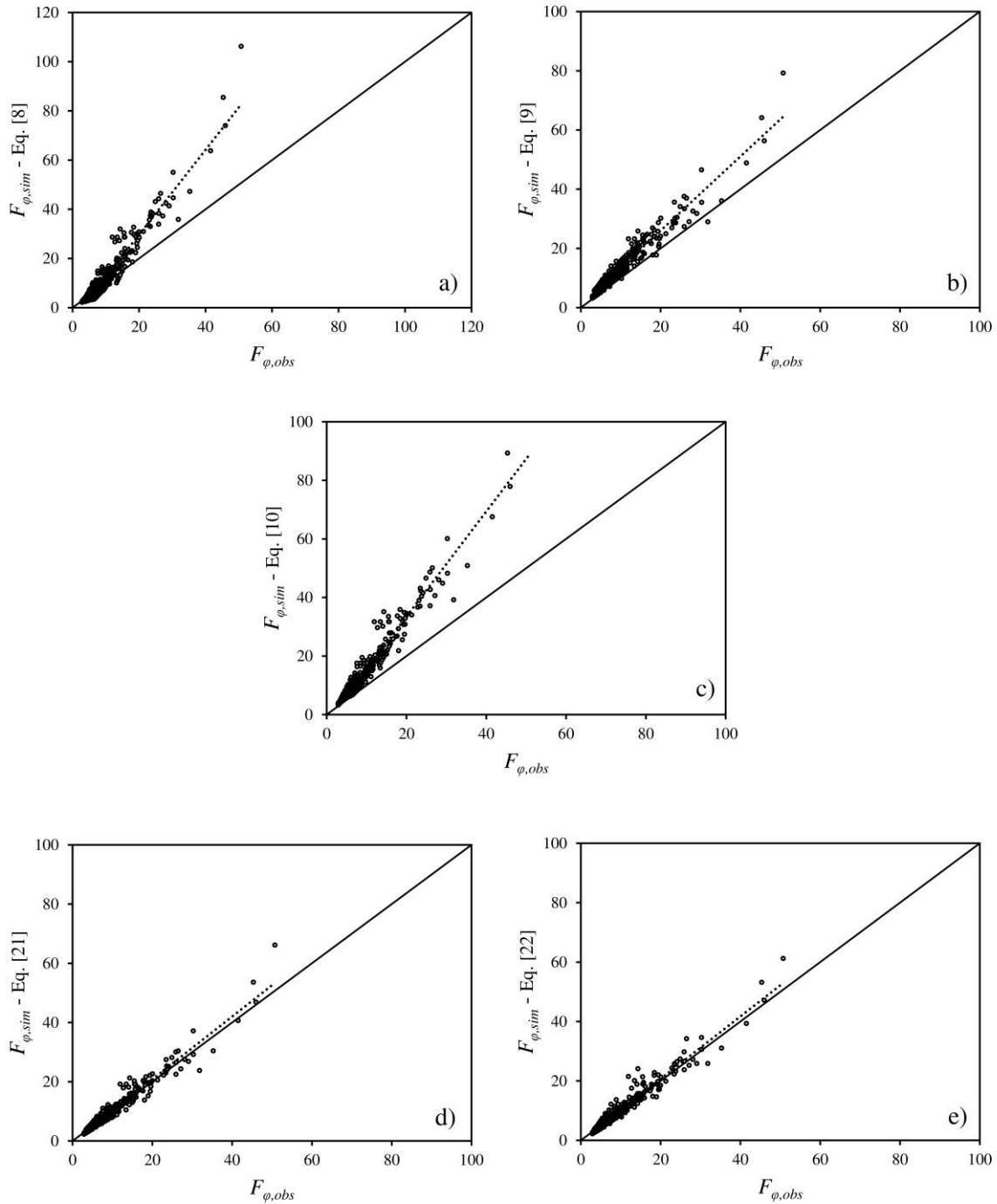


357

358 **Fig. 4.** Experimental  $F_\varphi \cdot (1 - \varphi^2 / \varphi^2)^{-1}$  values expressed as a function of  $Re_t$  (a) and  $Re_d$  (b); the  
 359 values of  $F_\varphi$  are calculated using Eq. (6).

360

361 Thus, Eqs. (21) and (22) allow values of  $F_\varphi$  exclusively to be estimated from geometric screen  
 362 parameters and airspeed. Figure 5 shows the experimentally observed  $F_{\varphi,obs}$  values calculated  
 363 using Eq. [6] compared with  $F_{\varphi,sim}$  values estimated using different models. For each screen  
 364 presented in Table 1, and at different values of air velocity ( $u$ ; ranging from 0.25 to 3.00 m s<sup>-1</sup>,  
 365 in intervals of +0.25 m s<sup>-1</sup>),  $F_\varphi$  values were estimated using previously published Eqs. (8)  
 366 (Hayama *et al.*, 2000), Eq. (9) (Pinker and Herbert, 1967) and Eq. (10) (Bailey *et al.*, 2003) and  
 367 using models proposed in this study Eqs. (21) (with porosity and Reynolds number based on  
 368 thread diameter  $Re_d$ ) and Eq. (22) (with porosity and Reynolds number based on the screen  
 369 thickness  $Re_t$ ). This figure shows that the  $F_{\varphi,sim}$  values estimated using Eqs. (21) and (22)  
 370 proposed in this study fit the observed  $F_{\varphi,obs}$  values better than the  $F_{\varphi,sim}$  values estimated using  
 371 Eqs. (8), (9) and (10).



372

373 **Fig. 5.**  $F_\varphi$  values estimated using Eqs. (8) (a), (9) (b), (10) (c), (21) (d) and (22) (e) compared  
 374 with experimental  $F_\varphi$  values (wind tunnel tests and with Eq. (6)).

375

376 Statistical analysis (Table 3) was also performed to identify the model for estimating  $F_{\varphi,sim}$  that  
 377 provides the values closest to the experimentally observed  $F_{\varphi,obs}$  values. Both models presented  
 378 in this study show significantly improved RMSD values with respect to models described in

379 the literature, reaching values of 18.9% (Eq. (21)) and 19.1% (Eq. (22)). In this case, the NSE  
380 statistic reached 0.9 for Eqs. (21) and (22), almost equal to the optimal value, which is also an  
381 improvement when compared with other models described in the literature (Eqs. (8), (9) and  
382 (10)). The PBIAS statistic had negative values in the five models, which indicated that all  
383 models overestimated the value of  $F_\phi$ , providing values higher than the experimentally  
384 observed values. However, the PBIAS values of the models presented in Eqs. (21) and (22)  
385 were very close to the optimal value of zero, in contrast to the other three models. The RSR  
386 values which can range from zero to  $\infty$ , were also improved by the models presented in Eqs.  
387 (21) and (22), with values closer to the optimal value of zero compared with the other models.  
388 The results show that the models developed in this study (Eqs. (21) and (22)) provided  $F_{\phi, sim}$   
389 values that fitted  $F_{\phi, obs}$  values better than the three models previously described in the literature.  
390 The first model, (Eq. (21)) required the same geometric parameters as the other models  
391 described in the literature; thus, to estimate  $F_\phi$  values, the values of porosity and thread diameter  
392 for the screens must be known. Conversely, to apply the second model (Eq. (22)), the values of  
393 porosity and thickness of the screen must be known. This may be an advantage over the other  
394 models because the process to measure the thickness may be simpler than measuring thread  
395 diameter when it is not possible to obtain microscopic images. Measurement of the thread  
396 diameter with a micrometer is difficult because of the non-cylindrical shape of the thread as  
397 consequence of deformations that are produced during screen manufacture. However, the  
398 measurement of the screen thickness with a micrometer is simple and more exact.  
399 Equation (8) by Hayama *et al.* (2000) was derived for wire nets; in this case, these authors  
400 worked with nine samples with 0.371, 0.504 and 0.778 porosity. The use of a metal wire net  
401 and only a small number of samples are possible reasons for Eq. (8) estimated  $F_\phi$  with  
402 significant deviation from the experimentally observed values in our screen samples.



403 Equation (9), developed by Pinker and Herbert (1967) was derived originally using a small  
 404 number of samples, with only 8 different samples of woven wire gauzes with porosity values  
 405 ranging from 0.3 to 0.7 m<sup>2</sup> m<sup>-2</sup>. These samples had a fabric structure similar to that of insect-  
 406 proof screens. However, the material used (metal wire) was different from the polyethylene  
 407 threads normally used today in insect-proof screens. These may be the reasons why this  
 408 equation predicts  $F_\phi$  values far from the experimentally observed values obtained here.

409 Equation (10), as used by Bailey *et al.* (2003), was derived by performing tests on 5 insect-  
 410 proof screens with 0.25, 0.45, 0.53, 0.66 and 0.68 porosity. These porosity values are  
 411 considerably different from the porosity values of the 35 screens used in this study, where  
 412 porosity ranged from 0.237 to 0.556 m<sup>2</sup> m<sup>-2</sup>. Of the 35 screens tested here, 29 had porosity  
 413 values lower than 0.45 m<sup>2</sup> m<sup>-2</sup>. This may explain why, that when using Eq. (10), the  $F_\phi$  values  
 414 assessed were also far from the experimentally observed values for our insect-proof screens.

415 Based on the findings of this study, it is recommended to apply an  $F_\phi$  prediction equation that  
 416 was derived using a sample of insect-proof screen as close to the example screen as possible.

417

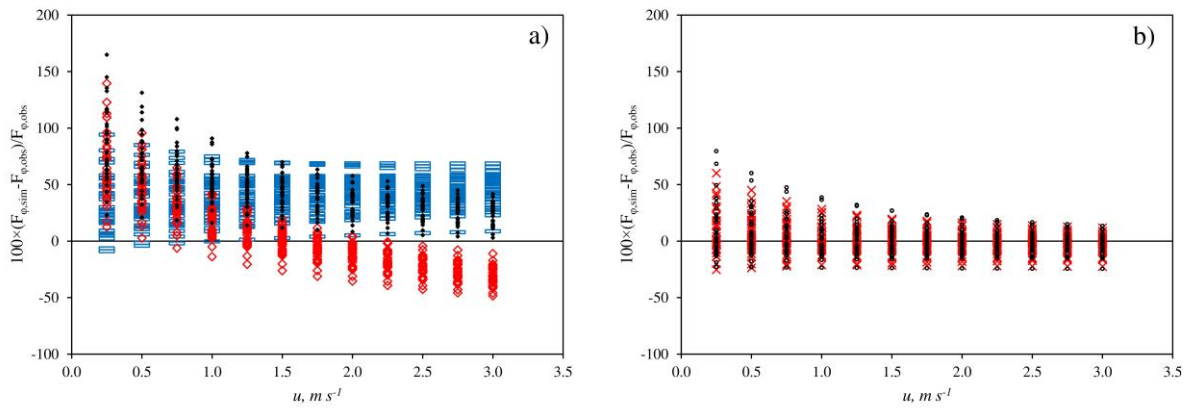
418 **Table 3.** Statistical parameters of the comparison between experimentally observed  $F_{\phi,obs}$   
 419 values and  $F_{\phi,sim}$  values estimated using equations proposed in the literature: Eq. (8) (Hayama  
 420 *et al.*, 2000), Eq. (9) (Pinker and Herbert, 1967) and Eq. (10) (Bailey *et al.*, 2003) along with  
 421 the equations developed in this study: namely Eqs. (21) and (22),  $R$ , the correlation coefficient;  
 422  $RMSD$ , root mean squared error;  $MD$ , bias;  $NSE$ , the Nash-Sutcliffe efficiency;  $PBIAS$ , percent  
 423 bias;  $RSR$ ,  $RMSD$ -observations standard deviation ratio.

	Eq. (8)	Eq. (9)	Eq. (10)	Eq. (21)	Eq. (22)
$R$	0.970	0.974	0.979	0.970	0.960
$RMSD$	6.3	4.4	8.0	1.7	1.7
$RMSD$ (%)	69.6	49.0	89.1	18.9	19.1
$MD$	-1.9	-3.7	-5.4	-0.2	-0.2
$NSE$	0.2	0.6	-0.3	0.9	0.9

<i>PBIAS</i>	-19.0	-36.1	-52.7	-1.6	-1.8
<i>RSR</i>	10.4	7.3	13.3	2.8	2.9

424

425 Conversely, in general, these equations for estimating  $F_\varphi$  values have a higher error in  
426 predicting  $F_\varphi$  for low air velocity values (Fig. 6). In the case of insect-proof screens, this is a  
427 serious drawback because the air velocity through these screens in a greenhouse with natural  
428 ventilation conditions is usually low, with maximum observed values of  $\sim 1.5 \text{ m s}^{-1}$  (Molina-  
429 Aiz *et al.*, 2009; López *et al.*, 2012). When using Eq. (8), 72% of the  $F_{\varphi, sim}$  values have an error  
430  $< 20\%$  of the  $F_{\varphi, obs}$  value. When using Eqs. (9) and (10), only 12% and 11% of the simulated  
431 values had an error  $< 20\%$  of the  $F_{\varphi, obs}$  value, respectively. However, when using Eqs. (21) and  
432 (22), proposed in this study, 94% and 92% of the  $F_{\varphi, sim}$  values had an error  $< 20\%$  of the  $F_{\varphi, obs}$   
433 value, respectively. Castellano *et al.* (2016) used equations similar to those used here to predict  
434 the aerodynamic characteristics of insect-proof screens with satisfactory results, but these  
435 authors performed tests in a wind tunnels using air velocities  $>4 \text{ m s}^{-1}$ .



436

437 **Fig. 6.** Error observed between  $F_{\varphi, obs}$  and  $F_{\varphi, sim}$  (%) as a function of the air velocity. (a)  $F_{\varphi, sim}$   
438 assessed using Eq. (8) ( $\diamond$ ) (Hayama *et al.*, 2000), Eq. (9) ( $=$ ) (Pinker and Herbert, 1967), [10]  
439 ( $\blacklozenge$ ) (Bailey *et al.*, 2003). (b)  $F_{\varphi, sim}$  assessed using Eqs. (21) ( $\times$ ) and (22) ( $\circ$ ).

440

441 **3.3. Models for estimating  $K_p$  and  $Y$  values of an insect-proof screen given their geometric**  
442 **characteristics**

443 Another option for the aerodynamic characterisation of an insect-proof screen is to use the  
444 screen permeability  $K_p$  and its inertial factor  $Y$ . Given these parameters, curves of pressure drop  
445 as a function of air velocity of a screen can be constructed by applying the modified Darcy's  
446 equation (Forchheimer, 1901), indicated in Eq. (1). These two parameters (Table 2) were  
447 determined in wind tunnel tests for the 35 study screens. Previously, Miguel (1998) presented  
448 two models for estimating the value of these two parameters from screen porosity (Eqs. (14)  
449 and (15)). The constants  $\beta$ ,  $A$  and  $B$ , in Eqs. (12) and (13) presented by Nield and Bejan (1999)  
450 were also determined statistically for the 35 screens analysed in this study. This led to the  
451 following equation for estimating the screen permeability  $K_p$  ( $\beta=2.0679$ ) with the thread  
452 diameter  $D_h$  [m] and porosity [ $\text{m}^2 \text{m}^{-2}$ ]:

453 
$$K_p = \frac{D_h^2 \varphi^3}{2.0679(1-\varphi)^2} + 3.8362 \times 10^{-10} \quad (24)$$

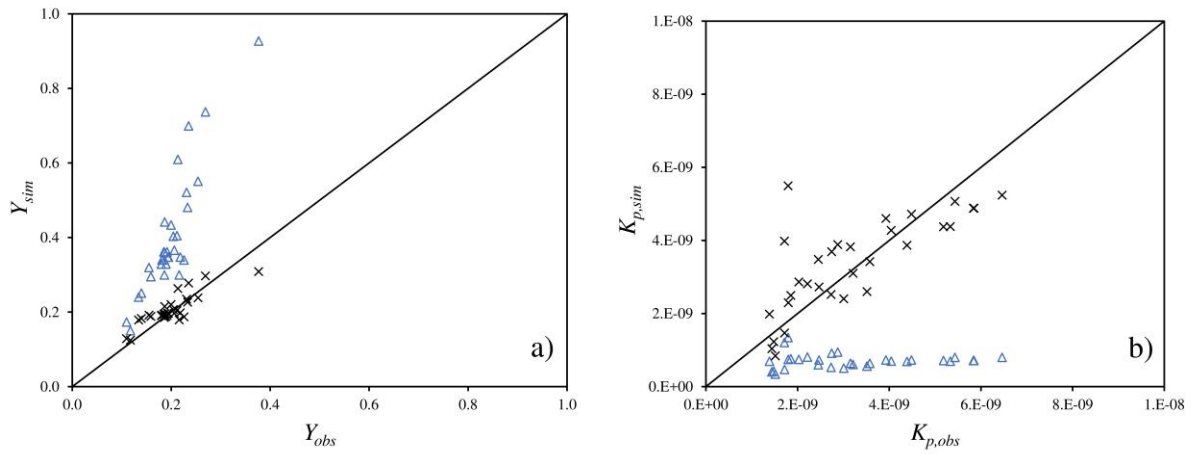
454 with 0.75 correlation coefficient, a 56.3%  $R^2$  and a  $p$ -value  $< 0.05$ . The following equation can  
455 be used to estimate the inertial factor  $Y$  of the screen ( $A = 0.0571195$  and  $B = 0.135966$ ) with  
456 the thread diameter [m] and the diameter of the inside circumference of the pore  $D_i$  [m]:

457 
$$Y = 0.0571195 + 0.135966 \frac{D_h}{D_i} \quad (25)$$

458 with a 0.76 correlation coefficient, a 58.1%  $R^2$  and a  $p$ -value  $< 0.0001$ .

459 Figure 7 shows the values of  $K_p$  (a) and  $Y$  (b) estimated using the Eqs. (14) and (15) as a function  
460 only of screen porosity proposed by Miguel (1998) and the Eqs. (24) and (25) obtained in this  
461 study in the way proposed by Nield and Bejan (1999) as a function of porosity and geometric  
462 characteristics of the screens, all of which are represented with respect to the experimentally  
463 observed values (Table 2). This figure shows that the inertial factor ( $Y$ ) values derived using  
464 the equations by Miguel (1998) are much higher than the experimentally observed values and  
465 that the permeability ( $K_p$ ) values are much lower than the experimentally observed values. With

466 Eqs. (24) and (25) derived from the geometric characteristics of the screen as suggested by  
467 Nield and Bejan (1999), the values similarly approximate the experimentally observed values.



468

469 **Fig. 7.** Simulated  $Y$  (a) and  $K_p$  (b) values compared to experimentally observed values. ( $\Delta$ ), Eqs.  
470 (14) and (15) proposed by Miguel (1998); ( $\times$ ), Eqs. (26) and (27) derived from the geometric  
471 characteristics of the screens.

472

473 Statistical analysis was also performed (Table 4) to identify the equation(s) for estimating the  
474  $K_p$  and  $Y$  values that provide the closest simulated values to the experimentally observed values.

475 The equations proposed by Miguel (1998) for estimating the  $K_p$  and  $Y$  values provide values  
476 very far from the experimentally observed values (Fig. 7 and Table 4). Therefore, in principle,

477 we advise against using these equations for several reasons. Miguel (1998) when developing

478 the model used only 14 screens. In addition, different types of screens were combined, 8 screens

479 with rectangular pores, similar to the screens used in this study, and 6 shade screens with

480 irregular pores made of a different type of fabric. Conversely, the equations of this study (Eqs.

481 (24) and (25)) were derived using more screens (35 in number), all of which were the same type

482 of fabric (weft and warp) with rectangular pores and with a narrow porosity range, from 0.237

483 to  $0.556 \text{ m}^2 \text{ m}^{-2}$ .

484 Based on the results, Eqs. (24) and (25) should be used to estimate  $K_p$  and  $Y$  values, in preference  
 485 to the equations proposed by Miguel (1998) as a function of only the screen porosity. Equations  
 486 (24) and (25), however, require knowledge of the thread diameter, the inner pore diameter and  
 487 the porosity. Also, the screen thickness  $e$  must also be known to apply Eq. (1) and thus construct  
 488 the pressure drop curve for the screen. In CFD software, a porous jump boundary conditions  
 489 can be used to model screens (Molina-Aiz *et al.*, 2017) and filters knowing values of  $K_p$ ,  $Y$  and  
 490  $e$ .

491

492 **Table 4.** Statistical parameters of the comparison between the experimentally observed  $K_{p,obs}$   
 493 and  $Y_{obs}$  values and the  $K_{p,sim}$  and  $Y_{sim}$  values estimated using the Eq. (14) and (15) (Miguel,  
 494 1998) proposed in the literature and Eqs. (24) and (25) based on the model proposed by Nield  
 495 and Bejan (1999); *RMSD*, root mean squared error; *MD*, *bias*; *NSE*, the Nash-Sutcliffe  
 496 efficiency; *PBIAS*, percent *bias*; *RSR*, *RMSD*-observations standard deviation ratio.

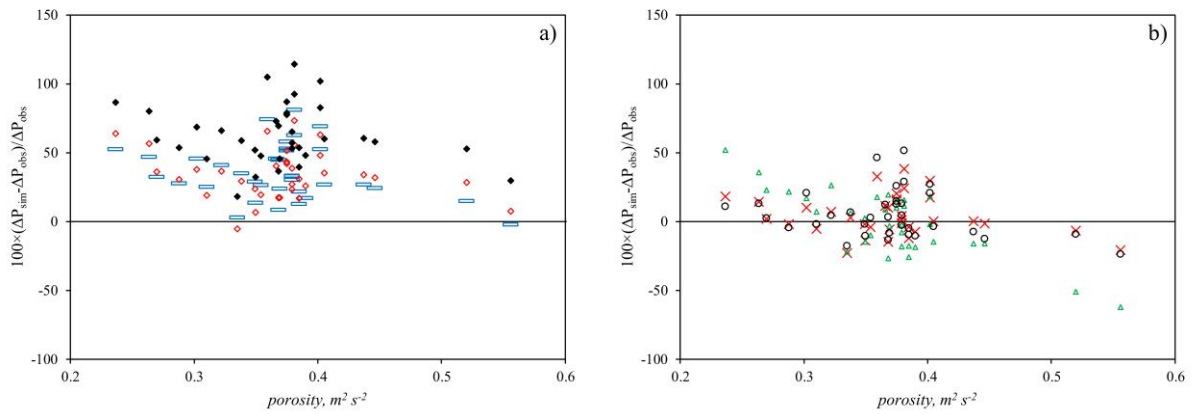
	$K_p$		$Y$	
	Eq. (14)	Eq. (24)	Eq. (15)	Eq. (25)
<i>R</i>	0.063	0.750	0.798	0.763
<i>RMSD</i>	$3.240 \times 10^{-9}$	$1.100 \times 10^{-9}$	0.220	0.025
<i>RMSD</i> (%)	92.9	31.5	107.9	12.1
<i>MD</i>	$7.058 \times 10^{-10}$	$3.487 \times 10^{-9}$	0.396	0.204
<i>NSE</i>	-2.8	0.6	-22.2	0.7
<i>PBIAS</i>	79.8	0.0	-87.1	-2.7
<i>RSR</i>	$1.933 \times 10^{-4}$	$6.561 \times 10^{-5}$	2.5	0.3

497

### 498 3.4. Effect of porosity in the pressure drop estimation with the different models for 499 calculate $F_\phi$ , $K_p$ and $Y$

500 For the 35 screens (Table 1) and with the different values of  $F_\phi$ , estimated using Eqs. (8) from  
 501 Hayama *et al.* (2000), (9) from Pinker and Herbert (1967) and (10) from Bailey *et al.* (2003)  
 502 along with Eqs. (21) and (22) derived from the present study, Eq. (5) can be used to obtain the  
 503 values of pressure drop as a function of air velocity. In the same way, with the values of  $K_p$  and

504  $Y$  estimated using Eqs. (14) and (15) (Miguel, 1998) and Eqs. (24) and (25) derived from this  
 505 work from the general equations used for porous media (Nield and Bejan, 1999), Eq. (1) can be  
 506 applied to obtain the values of pressure drop as a function of air velocity. Figure 8 shows the  
 507 error (%) between the pressure drops measured in the wind tunnel  $\Delta P_{obs}$  and those calculated  
 508 from the different equations  $\Delta P_{sim}$  as a function of the screen porosity. In Fig. 8a the error  
 509 obtained with previous models of the pressure drop coefficient  $F_\phi$  found in the literature is  
 510 shown. In Fig. 8b the error obtained with the different models (two for  $F_\phi$  and one using  $K_p$  and  
 511  $Y$ ) can be seen.



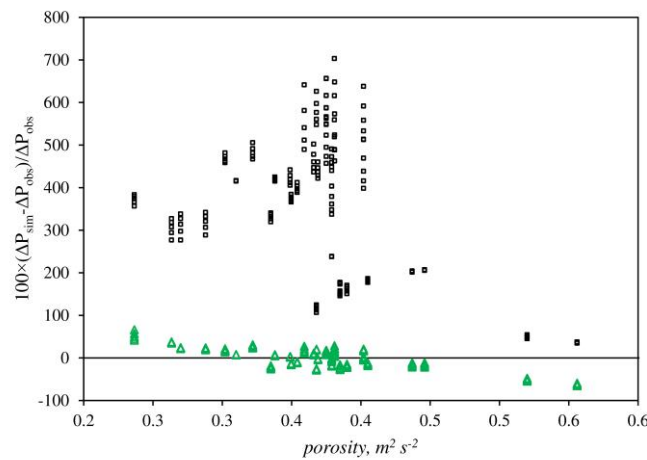
512  
 513 **Fig. 8.** Average error observed between  $\Delta P_{obs}$  and  $\Delta P_{sim}$  (%), for air velocities from 0.25 to 1.25  
 514 m s<sup>-1</sup>, as a function of the porosity of the screens. (a)  $\Delta P_{sim}$  estimated using Eq. (8) ( $\diamond$ ) from  
 515 Hayama *et al.* (2000), Eq. (9) ( $\square$ ) from Pinker and Herbert (1967), Eq. (10) ( $\blacklozenge$ ) from Bailey *et*  
 516 *al.* (2003). (b)  $\Delta P_{sim}$  estimated using Eq. (21) ( $\times$ ), Eq. (22) ( $\circ$ ) and Eqs. (24) and (25) ( $\triangle$ ).

517  
 518 In general, the pressure drops calculated from models of  $F_\phi$  obtained from the literature  
 519 overestimated the measured values for all porosities ranging from 0.24 to 0.56 (Fig. 8a). The  
 520 average errors for all porosities and velocities lower than 1.25 m s<sup>-1</sup> were 34.7% for Eq. (8),  
 521 from Hayama *et al.* (2000), 35.5% for Eq. (9) from Pinker and Herbert (1967) and 63.3% for  
 522 Eq. (10) from Bailey *et al.* (2003). However, the errors of the proposed models of  $F_\phi$ , derived  
 523 from porosity and the  $Re$  number obtained with the thread diameter (Eq. (21)) and the screen

524 thickness (Eq. (22)), appear to be uniform as a function of screen porosity, with average values  
 525 of 5.2% and 4.4%, respectively.

526 Using the values of aerodynamic properties of the screens (permeability  $K_p$  and inertial factor  
 527  $Y$ ) calculated with Eqs. (24) and (25), derived from porosity and two geometric characteristics  
 528 of the screen ( $D_h$  and  $D_i$ ), the average error was zero as consequence of an overestimation for  
 529 porosities lower than 0.3 and underestimation for porosities greater than 0.5 (Fig. 8b).

530 Figure 9 shows the error (%) between pressure drop measured experimentally in the wind tunnel  
 531  $\Delta P_{obs}$  and the pressure drop  $\Delta P_{sim}$  calculated with the values of  $K_p$  and  $Y$  estimated using Eqs.  
 532 (14) and (15) (Miguel, 1998) and Eqs. (24) and (25) derived from parameters presented by  
 533 Nield and Bejan (1999). In Fig. 9 it can be seen how the models presented by Miguel (1998)  
 534 provide a much higher error than the models obtain in this work for porosities  $< 0.5$ , and similar  
 535 errors for the two screens with porosities between 0.5 and 0.6.



536  
 537 **Fig. 9.** Error observed between pressure drop measured experimentally in the wind tunnel  $\Delta P_{obs}$   
 538 and calculated  $\Delta P_{sim}$  (%) as a function of the porosity of the screens.  $\Delta P_{sim}$  estimated using Eqs.  
 539 (14) and (15) proposed by Miguel (1998) ( $\square$ ) and Eqs. (26) and (27) derived from parameters  
 540 presented by Nield and Bejan (1999) ( $\Delta$ ), using air velocities  $u$  ranging from 0.25 to 1.25 m s<sup>-1</sup>  
 541 <sup>1</sup>, in +0.25 m s<sup>-1</sup> intervals).

542 The errors of Figs. 8 and 9 correspond to the models obtained with new insect-proof screens  
543 that have never been installed in a greenhouse. Once installed in the vents of the greenhouse,  
544 and with the passage of time, the screens deteriorate and this can cause the screen to become  
545 less rigid, increasing in thickness and reducing pressure drop (López *et al.*, 2018). On the other  
546 hand, the accumulation of dirt causes an important increase of  $F_\phi$  (between 16.5% and 61.2%;  
547 for  $u=1.0 \text{ m s}^{-1}$ ) and consequently increases in pressure drop (López *et al.*, 2018). These  
548 contrary effects of material ageing and dirt accumulation on the aerodynamic characteristics of  
549 screens must also be taken into account when applying the models described here.

#### 550 **4. Conclusions**

551 Different models for estimating the aerodynamic characteristics of insect-proof screens, with a  
552 weft and warp fabric, using one or more geometric parameters, were presented and validated in  
553 this study. These models were generated from data obtained from 35 insect-proof screens with  
554 porosity ranging from 0.237 to 0.556  $\text{m}^2 \text{ m}^{-2}$ .

555 Two options were validated for estimating the pressure drop coefficient  $F_\phi$ : the first model was  
556 based on the screen porosity and a Reynolds number based on thread diameter and the second  
557 model was based on the screen porosity and a Reynolds number based on thickness.

558 A third option was based on estimating the permeability  $K_p$  and the inertial factor  $Y$ . Two  
559 equations were obtained for estimating the value of  $K_p$  as a function of thread diameter  $D_h$  and  
560 porosity  $\phi$  and for estimating the value of  $Y$  as a function of thread diameter and the inner pore  
561 diameter  $D_i$ . These models have been shown to improve the accuracy of the previous models  
562 described in literature that overestimated the pressure drops for all porosities analysed.

563 Models for aerodynamic parameters of the insect-proof screens  $K_p$  and  $Y$  based in their  
564 geometric characteristics are very important to simulate the effect of this porous media in CFD  
565 studies. Knowledge of  $K_p$ ,  $Y$  and thickness allows to model insect-proof screens in CFD  
566 simulations as a thin membrane using the porous jump model applied to a face zone.



567 Future studies should be orientated towards the development of more robust models and it  
568 would be interesting to increase the porosity range and evaluate lower air velocities.

### 569 **5. Acknowledgements**

570 This work has been funded by the Spanish Ministry of Economy and Competitiveness and the  
571 European Regional Development Fund (ERDF) by means of the research grant AGL2015-  
572 68050-R. The authors wish to express their gratitude to the Research Centre CIAIMBITAL of  
573 the University of Almería (Spain) for their support throughout the development of this study.

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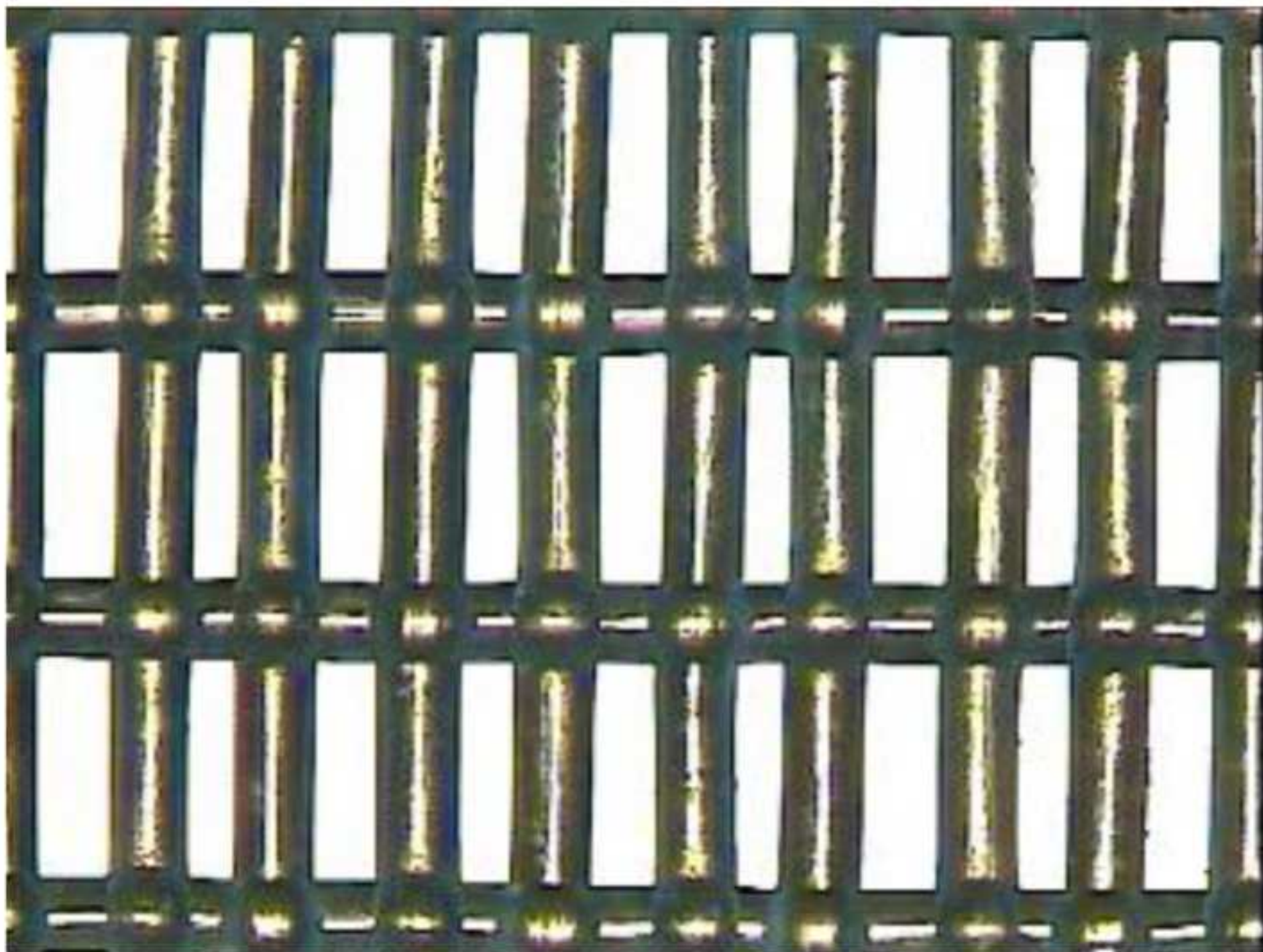
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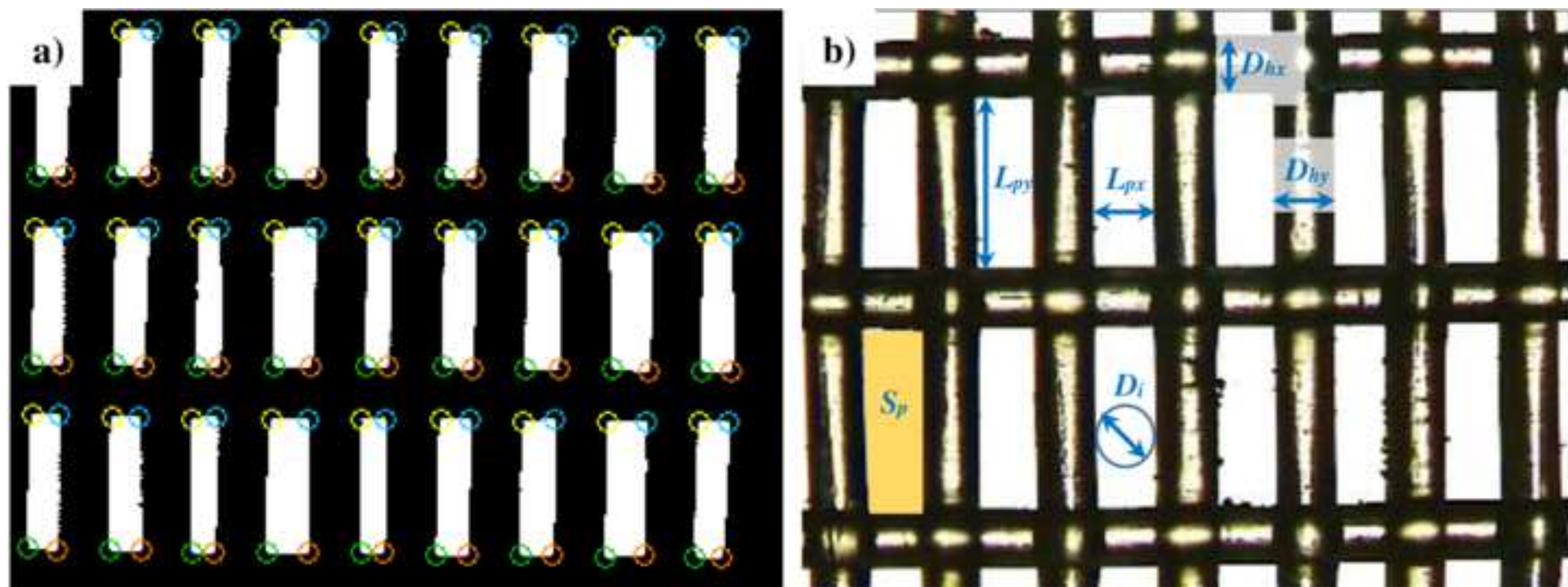
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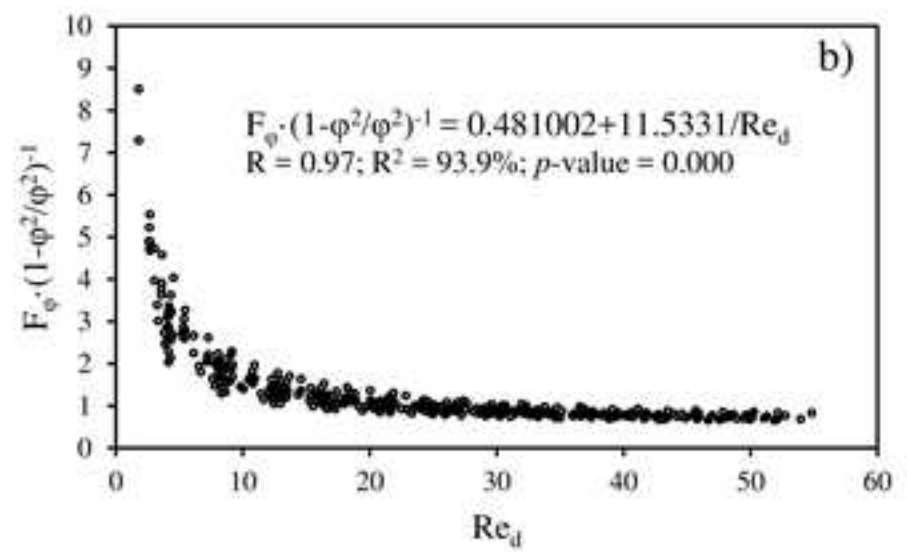
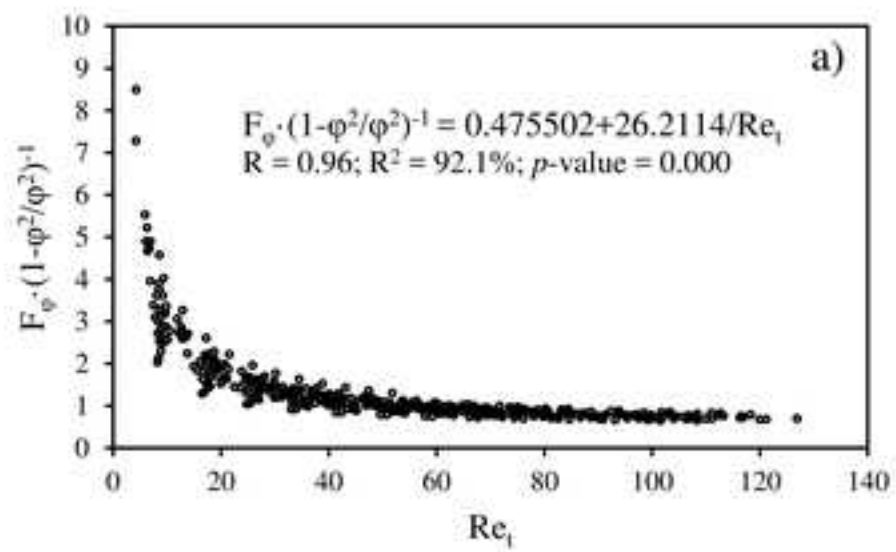
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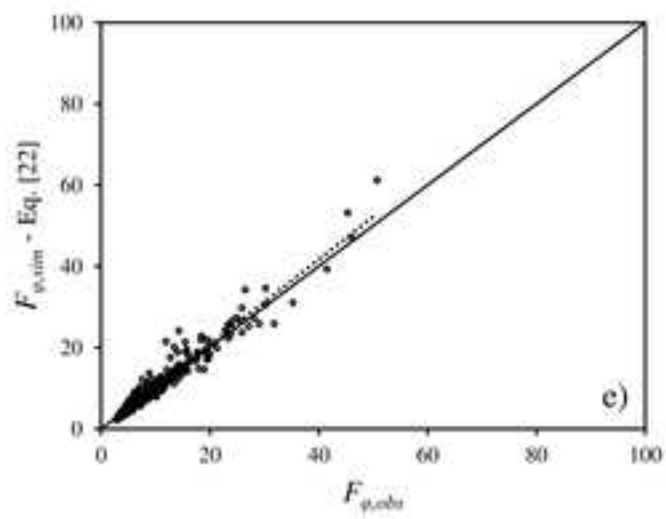
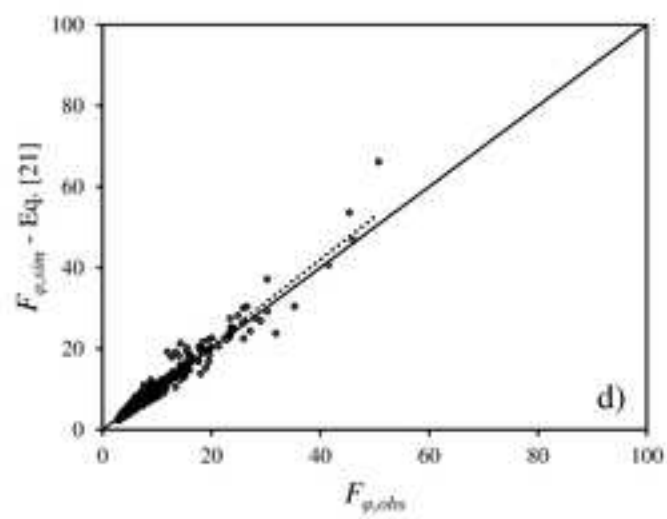
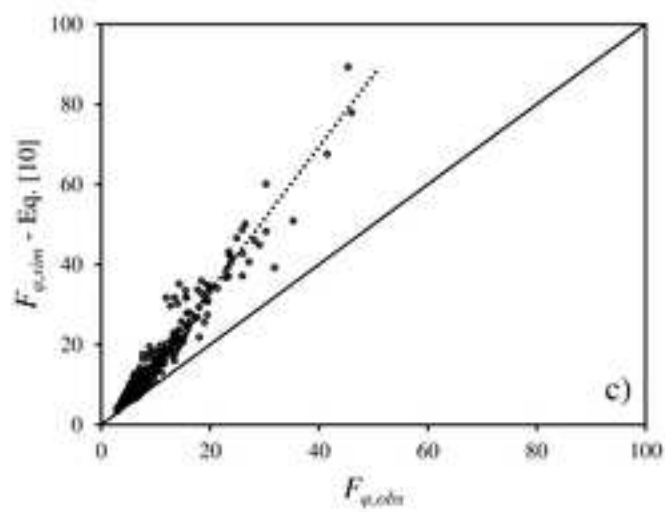
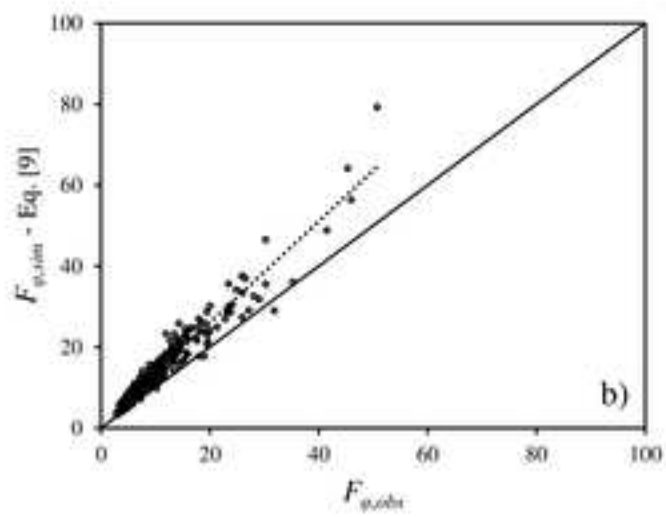
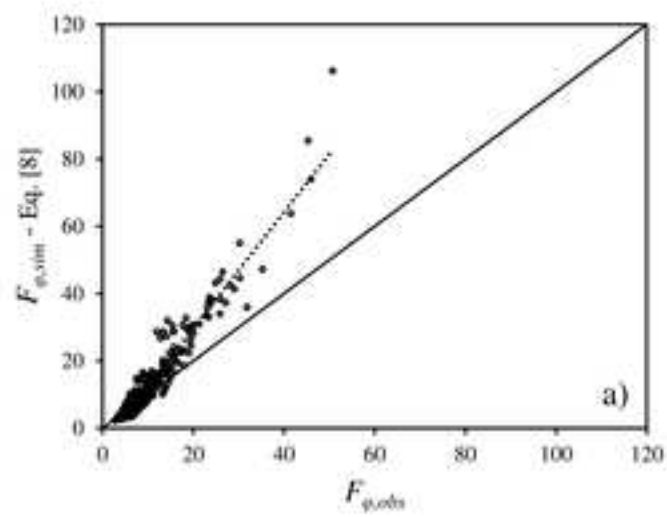


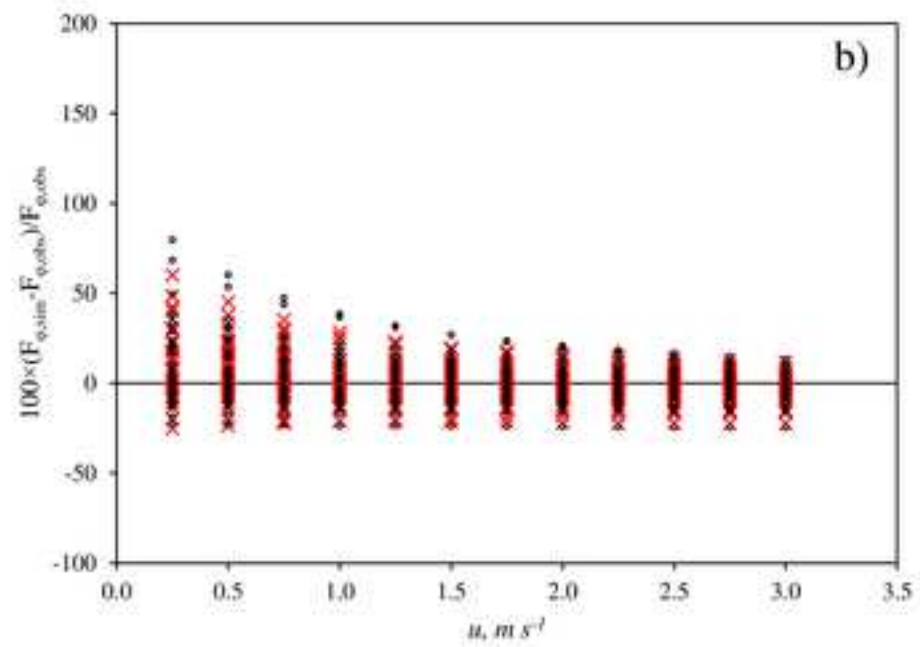
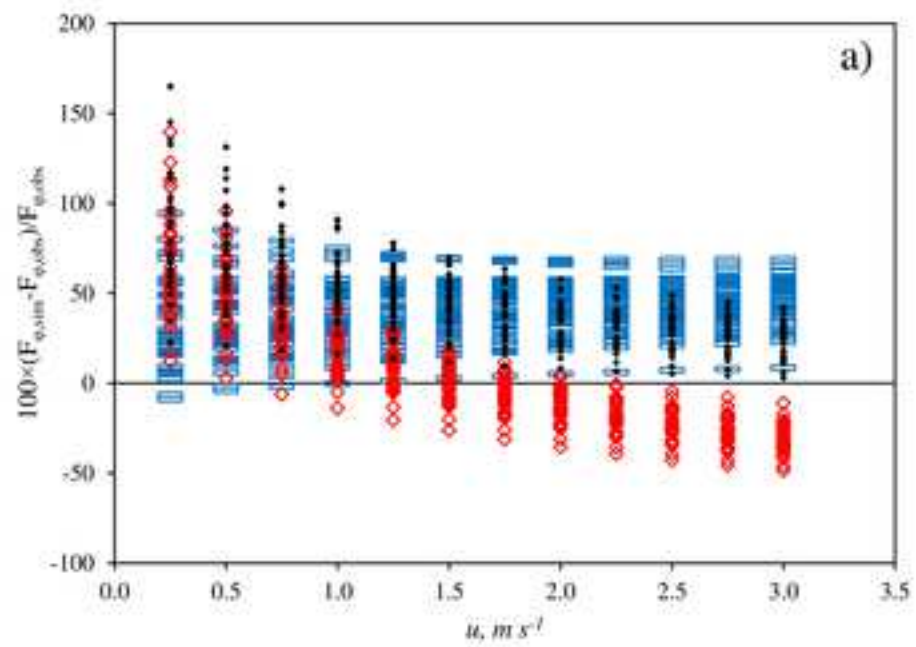


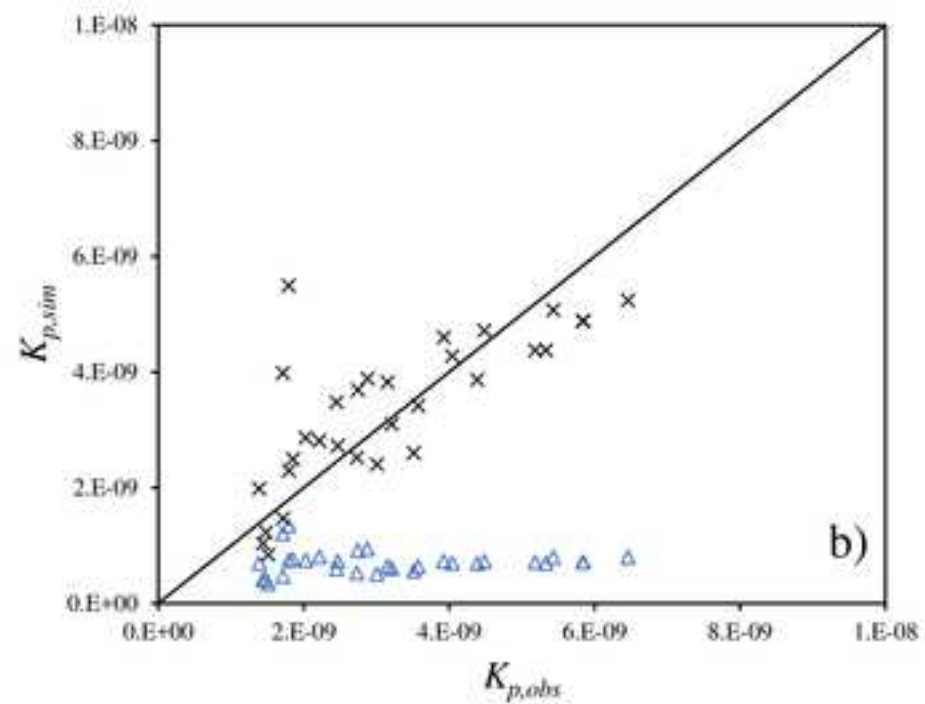
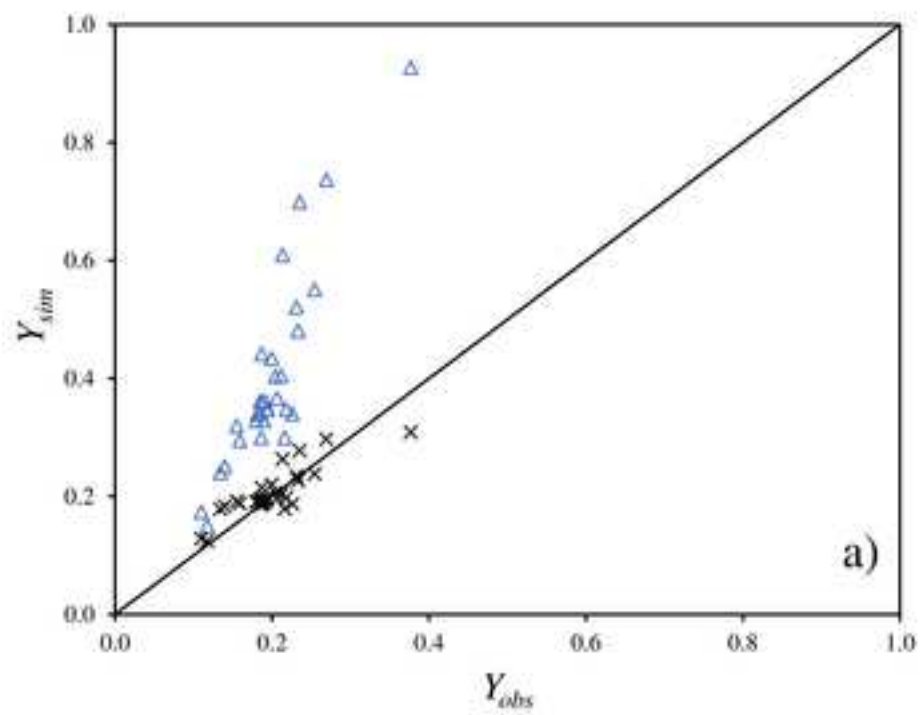




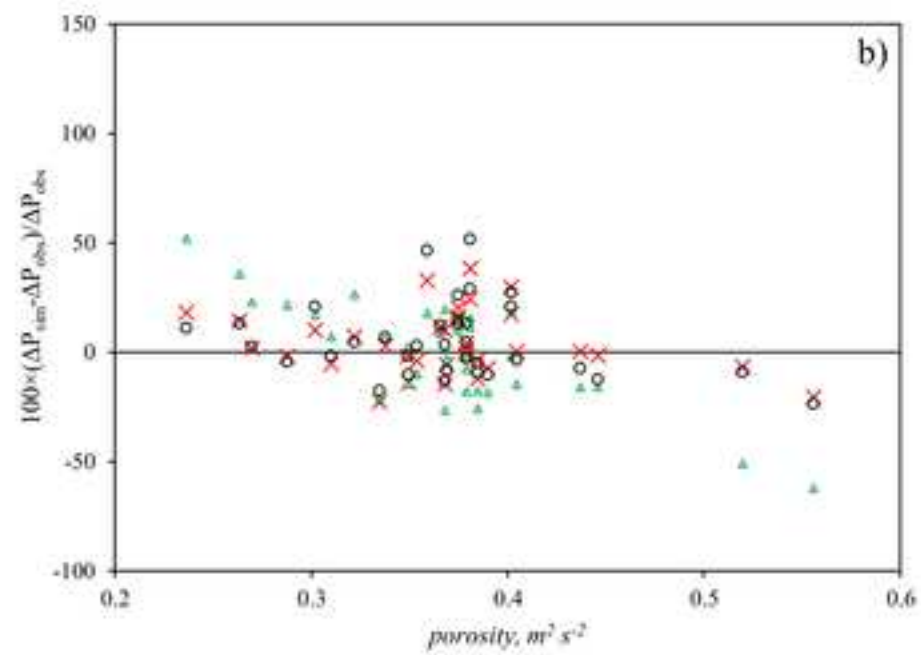
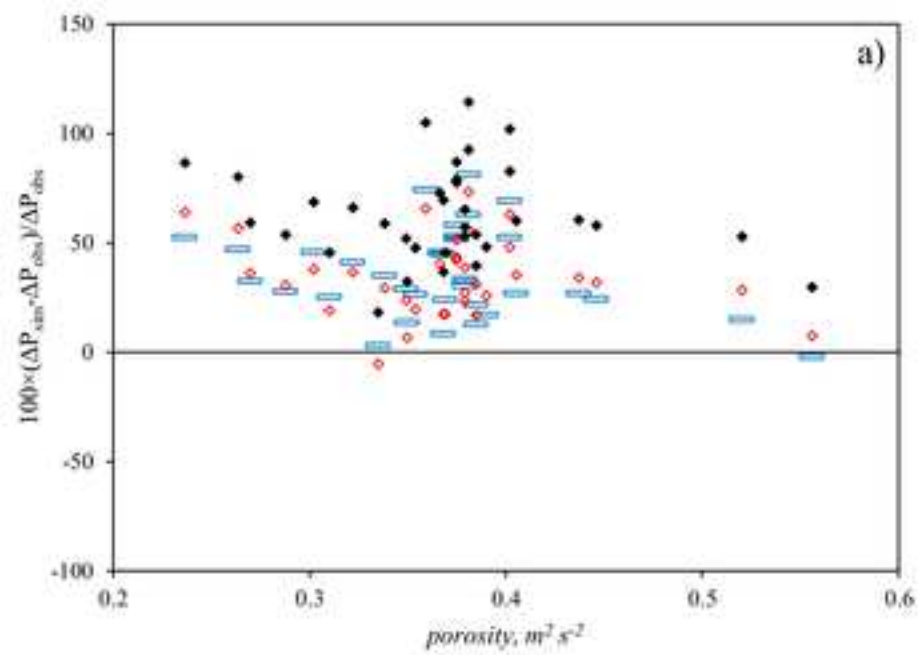


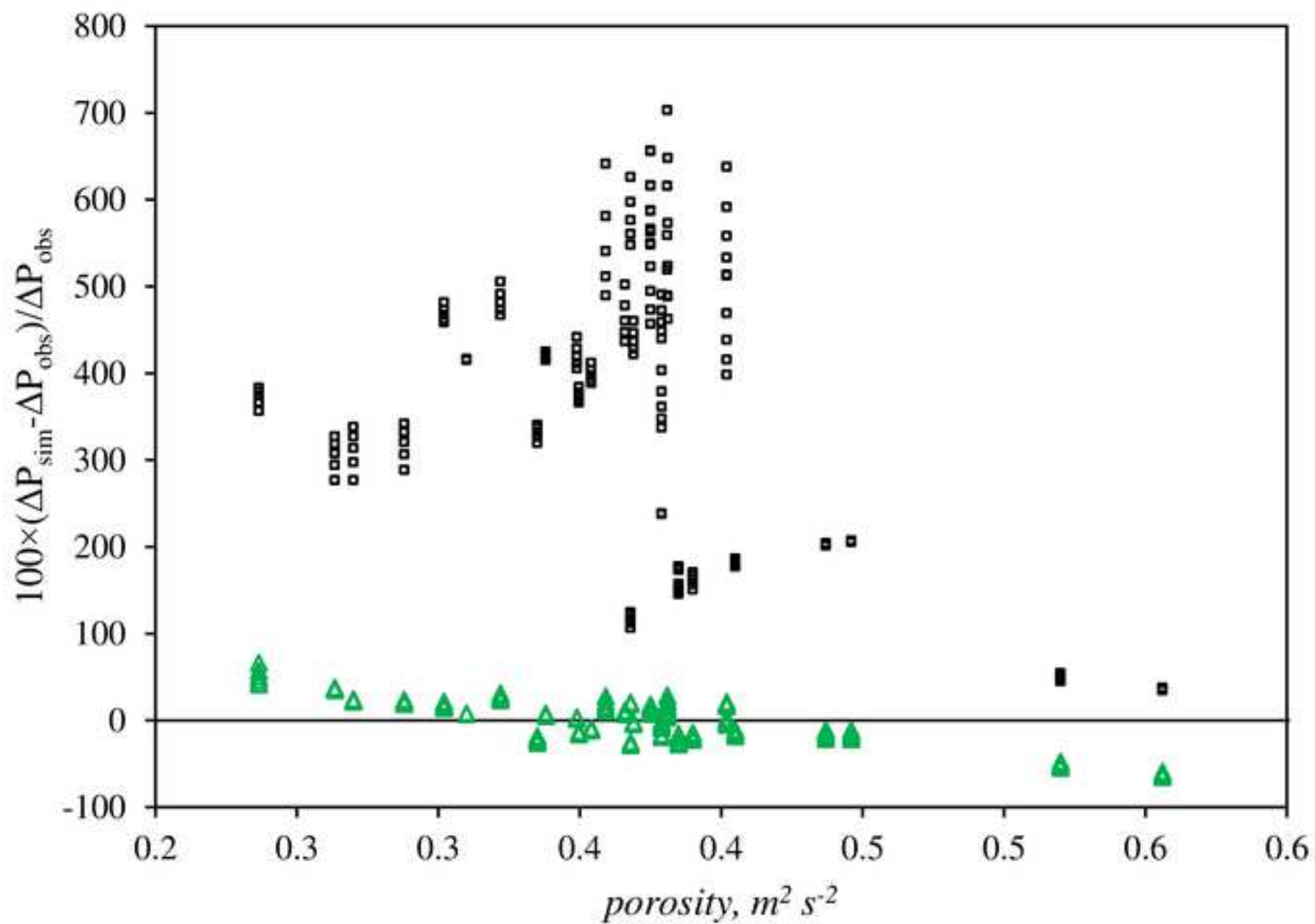














### Declaration of interests

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

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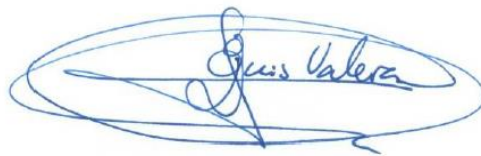
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